

Coarse large-eddy simulations in a transitional wake flow with flow models under location uncertainty

Pranav Chandramouli^{a,*}, Dominique Heitz^b, Sylvain Laizet^c, Etienne Mémin^a

^a*INRIA, Fluminance group, Campus universitaire de Beaulieu, F-35042 Rennes Cedex, France.*

^b*Irstea, UR OPAALE, F-35044 Rennes Cedex, France.*

^c*Department of Aeronautics, Imperial College London, London, United Kingdom.*

Abstract

The focus of this paper is to perform coarse-grid large eddy simulation (LES) using recently developed sub-grid scale (SGS) models of cylinder wake flow at Reynolds number (Re) of 3900. As we approach coarser resolutions, a drop in accuracy is noted for all LES models but more importantly, the numerical stability of classical models is called into question. The objective is to identify a statistically accurate, stable sub-grid scale (SGS) model for this transitional flow at a coarse resolution. The proposed new models under location uncertainty (MULU) are applied in a deterministic coarse LES context and the statistical results are compared with variants of the Smagorinsky model and various reference data-sets (both experimental and Direct Numerical Simulation (DNS)). MULU are shown to better estimate statistics for coarse resolution (at 0.46% the cost of a DNS) while being numerically stable. The performance of the MULU is studied through statistical comparisons, energy spectra, and sub-grid scale (SGS) contributions. The physics behind the MULU are characterised and explored using divergence and curl functions. The additional terms present (velocity bias) in the MULU are shown to improve model performance. The spanwise periodicity observed at low Reynolds is achieved at this moderate Reynolds number through the curl function, in coherence with the birth of streamwise vortices.

Keywords:

*Corresponding author.

Email address: pranav.chandramouli@inria.fr (Pranav Chandramouli)

1. Introduction

Cylinder wake flow has been studied extensively starting with the experimental works of Townsend (1; 2) to the numerical works of Kravchenko and others (3; 4; 5). The flow exhibits a strong dependence on Reynolds number $Re = UD/\nu$, where U is the inflow velocity, D is the cylinder diameter and ν is the kinematic viscosity of the fluid. Beyond a critical Reynolds number $Re \sim 40$ the wake becomes unstable leading to the well known von Kármán vortex street. The eddy formation remains laminar until gradually being replaced by turbulent vortex shedding at higher Re . The shear layers remain laminar until $Re \sim 400$ beyond which the transition to turbulence takes place up to $Re = 10^5$ - this regime, referred to as the sub-critical regime, is the focus of this paper.

The transitional nature of the wake flow in the sub-critical regime, especially in the shear layers is a challenging problem for turbulence modelling and hence has attracted a lot of attention. The fragile stability of the shear layers leads to more or less delayed roll-up into von Kármán vortices and shorter or longer vortex formation regions. As a consequence significant discrepancies have been observed in near wake quantities both for numerical simulations (6) and experiments (7).

Within the sub-critical regime, 3900 has been established as a benchmark Re . The study of (7) provides accurate experimental data-set showing good agreement with previous numerical studies contrary to early experimental datasets (8). The early experiments of Lourenco and Shih (9) obtained a V-shaped mean streamwise velocity profile in the near wake contrary to the U-shaped profile obtained by (8). The discrepancy was attributed to inaccuracies in the experiment - a fact confirmed by the studies of (10) and (4). Parnaudeau *et al.*'s (7) experimental database, which obtains the U-shaped mean profile in the near wake, is thus becoming useful for numerical validation studies. With increasing computation power, the LES data sets at $Re = 3900$ have been further augmented with DNS studies performed by (6).

The transitional nature of the flow combined with the availability of validated experimental and numerical data-sets at $Re = 3900$ makes this an ideal flow for model development and comparison. The LES model parametrisation controls the turbulent dissipation. A good SGS model should ensure

suitable dissipation mechanism. Standard Smagorinsky model (11) based on an equilibrium between turbulence production and dissipation, has a tendency to overestimate dissipation in general (12). In transitional flows, where the dissipation is weak, such a SGS model leads to laminar regimes, for example, in the shear layers for cylinder wake. Different modifications of the model have been proposed to correct this behaviour. As addressed by (12), who introduced relevant improvements, the model coefficients exhibit a strong dependency both on the ratio between the integral length scale and the LES filter width, and on the ratio between the LES filter width and the Kolmogorov scale. In this context of SGS models, coarse LES remains a challenging issue.

The motivation for coarse LES is dominated by the general interest towards reduced computational cost which could pave the way for performing higher Re simulations, sensitivity analyses, and Data Assimilation (DA) studies. DA has gathered a lot of focus recently with the works of (13), (14), and (15) but still remains limited by computational requirement.

With the focus on coarse resolution, this study analyses the performance of LES models for transitional wake flow at Re 3900. The models under location uncertainty (16; 17) are analysed in depth for their performance at a coarse resolution and compared with classical models. The models are so called as the equations are derived assuming that the location of a fluid parcel is known only up to a random noise i.e. location uncertainty. Within this reformulation of the Navier-Stokes equations, the contributions of the subgrid scale random component is split into an inhomogeneous turbulent diffusion and a velocity bias which corrects the advection due to resolved velocity field. Such a scheme has been shown to perform well on the Taylor Green vortex flow (18) at Reynolds number of 1600, 3000, and 5000. The new scheme was shown to outperform the established dynamic Smagorinsky model especially at higher Re. However, this flow is associated to an almost isotropic turbulence and no comparison with data is possible (as it is a pure numerical flow). Here we wish to assess the model skills with respect to more complex situations (with laminar, transient and turbulent areas) and coarse resolution grids. We provide also a physical analysis of the solutions computed and compare them with classical LES schemes and experimental data. Although the models are applied to a specific Reynolds number, the nature of the flow generalises the applicability of the results to a wide range of Reynolds number from $10^3 - 10^5$, i.e. up to the pivotal point where the transition into turbulence of the boundary layer starts at the wall of the cylinder. The goal

is to show the ability of such new LES approaches for simulation at coarse resolution of a wake flow in the subcritical regime. Note that recently for the same flow configuration, (19) have derived the Mulu in a reduced-order form using Proper Orthogonal Decomposition (POD), successfully providing physical interpretations of the local corrective advection and diffusion terms. The authors showed that the near wake regions like the pivotal zone of the shear layers rolling into vortices are key players into the modelling of the action of small-scale unresolved flow on the resolved flow .

In the following, we will show that the Mulu are able to capture, in the context of coarse simulation, the essential physical mechanisms of the transitional very near wake flow. This is due to the split of the SGS contribution into directional dissipation and velocity bias. The next section elaborates on the various SGS models analysed in this study followed by a section on the flow configuration and numerical methods used. A comparison of the elaborated models and the associated physics is provided in the results section. Finally, a section of concluding remarks follows.

2. Models under location uncertainty

General classical models such as Smagorinsky or Wall-Adaptive Local Eddy (WALE) viscosity model proceed through a deterministic approach towards modelling the SGS dissipation tensor. However, (16) suggests a stochastic approach towards modelling the SGS contributions in the Navier-Stokes (NS) equation. Building a stochastic NS formulation can be achieved via various methods. The simplest way consists in considering an additional additive random forcing (20). Other modelling considered the introduction of fluctuations in the subgrid models (21; 22). Also, in the wake of Kraichnan's work (23) another choice consisted in closing the large-scale flow in the Fourier space from a Langevin equation (24; 25; 26). Lagrangian models based on Langevin equation in the physical space have been also successfully proposed for turbulent dispersion (27) or for probability density function (PDF) modelling of turbulent flows (28; 29; 30). These attractive models for particle based representation of turbulent flows are nevertheless not well suited to a large-scale Eulerian modelling.

In this work we rely on a different stochastic framework of the NS equation recently derived from the splitting of the Lagrangian velocity into a smooth component and a highly oscillating random velocity component (i.e. the

uncertainty in the parcel location expressed as velocity) (16):

$$\frac{d\mathbf{X}_t}{dt} = \mathbf{u}(\mathbf{X}_t, t) + \sigma(\mathbf{X}_t, t)\dot{\mathbf{B}}$$

The first term, on the right-hand side represents the large-scale smooth velocity component, while the second term is the small-scale component. This latter term is a random field defined from a Brownian term function $\dot{\mathbf{B}} = d\mathbf{B}/dt$ and a diffusion tensor σ . The small-scale component is rapidly decorrelating at the resolved time scale with spatial correlations (which might be inhomogeneous and non stationary) fixed through the diffusion tensor. It is associated with a covariance tensor:

$$Q_{ij}(\mathbf{x}, \mathbf{y}, t, t') = \mathbb{E}((\sigma(\mathbf{x}, t)d\mathbf{B}_t)(\sigma(\mathbf{y}, t')d\mathbf{B}_{t'})^T) = c_{ij}(\mathbf{x}, \mathbf{y}, t)\delta(t - t')dt. \quad (1)$$

In the following the diagonal of the covariance tensor, termed here as the variance tensor, plays a central role; it is denoted as $\mathbf{a}(\mathbf{x}) \triangleq c(\mathbf{x}, t)$. This tensor is a (3×3) symmetric positive definite matrix with dimension of a viscosity in $\text{m}^2 \text{s}^{-1}$.

With such a decomposition, the rate of change of a scalar within a material volume, is given through a stochastic representation of the Reynolds Transport Theorem (RTT) (16; 17). For an incompressible small-scale random component ($\nabla \cdot \sigma = 0$) the RTT has the following expression:

$$d \int_{V(t)} q = \int_{V(t)} \left(d_t q + (\nabla \cdot (q \tilde{\mathbf{u}}) - \frac{1}{2} \sum_{i,j=1}^d \partial_{x_i} (a_{ij} \partial_{x_j} q)) dt + \nabla q \cdot \sigma d\mathbf{B}_t \right) d\mathbf{x}. \quad (2)$$

where the effective advection $\tilde{\mathbf{u}}$ is defined as:

$$\tilde{\mathbf{u}} = \mathbf{u} - \frac{1}{2} \nabla \cdot \mathbf{a}. \quad (3)$$

The first term on the right-hand side represents the variation of quantity q with respect to time: $d_t q = q(\mathbf{x}, t+dt) - q(\mathbf{x}, t)$. It is similar to the temporal derivative. It is important here to quote that q is a non differentiable random function that depends among other things on the particles driven by the Brownian component and flowing through a given location. The second term on the right-hand side stands for the scalar transport by the large-scale velocity. However, it can be noticed that this scalar advection is not

purely a function of the large-scale velocity. Indeed, the large-scale velocity is here affected by the inhomogeneity of the small-scale component through a modified large-scale advection (henceforth termed as velocity bias $\tilde{\mathbf{u}}$), where the effect of the fluctuating component is taken into account via the small-scale velocity auto-correlations $\mathbf{a} = (\boldsymbol{\sigma}\boldsymbol{\sigma}^T)$. A similar modification of the large-scale velocity was also suggested in random walks Langevin models by (31) who studied various stochastic models for particle dispersion - they concluded that an artificially introduced bias velocity to counter particle drift was necessary to optimise the models for a given flow. In the framework of modelling under location uncertainty, this term appears automatically. The third term in the stochastic RTT corresponds to a diffusion contribution due to the small-scale components. This can be compared with the SGS dissipation term in LES Modelling. This dissipation term corresponds to a generalization of the classical SGS dissipation term, which ensues in the usual context from the Reynolds decomposition and the Boussinesq's eddy viscosity assumption. Here it figures the mixing effect exerted by the small-scale component on the large-scale component. Despite originating from a very different construction, in the following, for ease of reference, we keep designating this term as the SGS contribution. The final term in the equation is the direct scalar advection by the small-scale noise.

It should be noted that the RTT corresponds to the differential of the composition of two stochastic processes. The Ito formulae, which is restricted to deterministic functions of a stochastic process, does not apply here. An extended formulae known as Ito-Wentzell (or generalized Ito) formulae must be used instead (32).

Using the Stochastic RTT, the large-scale flow conservation equations can be derived (for the full derivation please refer to (16; 17)). The final conservation equations are presented below:

Mass conservation:

$$d_t\rho + \nabla \cdot (\rho\tilde{\mathbf{u}})dt + \nabla\rho \cdot \boldsymbol{\sigma} d\mathbf{B}_t = \frac{1}{2}\nabla \cdot (\mathbf{a}\nabla\rho)dt, \quad (4)$$

which simplifies to the following constraints for an incompressible fluid:

$$\nabla \cdot \boldsymbol{\sigma} = 0, \quad \nabla \cdot \tilde{\mathbf{u}} = 0, \quad (5)$$

The first constraint maintains a divergence free small-scale velocity field, while the second imposes the same for the large smooth effective component.

We observe that the large-scale component, $\mathbf{u}$, is allowed to be diverging, with a divergence given by $\nabla \cdot \nabla \cdot \mathbf{a}$. As we shall see, this value is in practice quite low. This weak incompressibility constraint results in a modified pressure computation, which is numerically not difficult to handle. Imposing instead a stronger incompressibility constraint on $\mathbf{u}$ introduces an additional cumbersome constraint on the variance tensor ($\nabla \cdot \nabla \cdot \mathbf{a} = 0$). In this work we will rely on the weak form of the incompressibility constraint. The large-scale momentum equation boils down to a large-scale deterministic equation after separation between the bounded variation terms (i.e. terms in "dt") and the Brownian terms, which is rigorously authorized - due to uniqueness of this decomposition.

Momentum conservation:

$$\left(\partial_t \mathbf{u} + \mathbf{u} \nabla^T (\mathbf{u} - \frac{1}{2} \nabla \cdot \mathbf{a}) - \frac{1}{2} \sum_{ij} \partial_{x_i} (a_{ij} \partial_{x_j} \mathbf{u}) \right) \rho = \rho \mathbf{g} - \nabla p + \mu \Delta \mathbf{u}. \quad (6)$$

Similar to the deterministic version of the NS equation, we have the flow material derivative, the forces, and viscous dissipation. The difference lies in the modification of the advection which includes the velocity bias and the presence of the dissipation term which can be compared with the SGS term present in the filtered NS equation. Both the additional terms present in the stochastic version are computed via the auto-correlation tensor $\mathbf{a}$. Thus to perform a LES, one needs to model, either directly or through the small-scale noise, the auto-correlation tensor. Two methodologies can be envisaged towards this: the first would be to model the stochastic small-scale noise ($\sigma(\mathbf{X}_t, t) \dot{\mathbf{B}}$) and thus evaluate the auto-correlation tensor. We term such an approach as purely 'stochastic LES'. The second method would be to model the auto-correlation tensor directly as it encompasses the total contribution of the small scales. This method can be viewed as a form of 'deterministic LES' using stochastically derived conservation equations and this is the approach followed in this paper. The crux of the 'deterministic LES' approach thus revolves around the characterisation of the auto-correlation tensor. The small-scale noise is considered subsumed within the mesh and is not defined explicitly.

This opens up various possibilities for turbulence modelling. The specification of the variance tensor $\mathbf{a}$ can be performed through an empirical local velocity fluctuation variance times a decorrelation time, or by physical models/approximations or using experimental measurements. The options

explored in this study include physical approximation based models and empirical local variance based models as described below. Note that this derivation can be applied to any flow model. For instance, such a modelling has been successfully applied to derive stochastic large-scale representation of geophysical flows by (17; 33; 34).

A similar stochastic framework arising also from a decomposition of the Lagrangian velocity has been proposed in (35) and analysed in (36) and (37). This framework leads to enstrophy conservation whereas the formulation under location uncertainty conserves the kinetic energy of a transported scalar (17).

2.0.1. Physical approximation based models:

Smagorinsky's work on atmospheric flows and the corresponding model developement is considered to be the pioneering work on LES modelling (11). Based on Boussinesq's eddy viscosity hypothesis which postulates that the momentum transfer caused by turbulent eddies can be modelled by an eddy viscosity (ν_t) combined with Prandtl's mixing length hypothesis he developed a model (Smag) for characterising the SGS dissipation.

$$\nu_t = C\|\mathbf{S}\|, \quad (7)$$

$$\boldsymbol{\tau} = C\|\mathbf{S}\|\mathbf{S}, \quad (8)$$

where $\boldsymbol{\tau}$ stands for the SGS stress tensor, C is the Smagorinsky coefficient defined as $(C_s\Delta)^2$, where Δ is the LES filter width, $\|\mathbf{S}\| = \frac{1}{2}[\sum_{ij}(\partial_{x_i}u^j + \partial_{x_j}u^i)^2]^{\frac{1}{2}}$ is the Frobenius norm of the rate of strain tensor, and

$$S_{ij} = \frac{1}{2}\left(\frac{\partial \bar{u}_i}{\partial x_j} + \frac{\partial \bar{u}_j}{\partial x_i}\right). \quad (9)$$

Similar to Smagorinsky's eddy viscosity model, the variance tensor for the formulation under location uncertainty can also be specified using the strain rate tensor. Termed in the following as the Stochastic Smagorinsky model (StSm), it specifies the variance tensor similar to the eddy viscosity in the Classical Smagorinsky model:

$$\boldsymbol{a}(\mathbf{x}, t) = C\|\mathbf{S}\|\mathbb{I}_3, \quad (10)$$

where $\mathbb{I}_3$ stands for 3×3 identity matrix and C is the Smagorinsky coefficient. The equivalency between the two models can be obtained in the following case (as shown by (16)):

The SGS contribution (effective advection and SGS dissipation) for the StSm model is:

$$\begin{aligned}
u_j \partial_{x_j} (\partial_{x_j} a_{kj}) + \sum_{ij} \partial_{x_i} (a_{ij} \partial_{x_j} u_k) &= u_j \partial_{x_j} (\partial_{x_j} \|\mathbf{S}\| \delta_{kj}) + \sum_{ij} \partial_{x_i} (\|\mathbf{S}\| \delta_{ij} \partial_{x_j} u_k), \\
&= u_k \Delta \|\mathbf{S}\| + \|\mathbf{S}\| \Delta u_k + \sum_j \partial_{x_j} \|\mathbf{S}\| \partial_{x_j} u_k,
\end{aligned} \tag{11}$$

and the SGS contribution for Smagorinsky model ($\nabla \cdot \boldsymbol{\tau}$) is:

$$\begin{aligned}
\nabla \cdot \boldsymbol{\tau} &= \sum_j \partial_{x_j} (\|\mathbf{S}\| \mathbf{S}), \\
&= \sum_j \partial_{x_j} (\|\mathbf{S}\| (\partial_{x_j} u_k + \partial_{x_k} u_j)), \\
&= \sum_j \partial_{x_j} \|\mathbf{S}\| \partial_{x_j} u_k + \partial_{x_j} \|\mathbf{S}\| \partial_{x_k} u_j + \|\mathbf{S}\| \Delta u_k.
\end{aligned} \tag{12}$$

An equivalency can be drawn between the two models by adding $\sum_j \partial_{x_j} \|\mathbf{S}\| \partial_{x_k} u_j - u_k \Delta \|\mathbf{S}\|$ to the StSm model. The additional term may also be written as:

$$\partial_{x_k} \sum_j \partial_{x_j} (\|\mathbf{S}\|) u_j - \sum_j \partial_{x_j} \partial_{x_k} (\|\mathbf{S}\|) u_j - u_k \Delta \|\mathbf{S}\|, \tag{13}$$

where the first term represents a velocity gradient which can be included within a modified pressure term as is employed for Smagorinsky model. The other two terms can be neglected for smooth enough strain rate tensor function. For smooth deformations both models are equivalent in terms of dissipation. It is important to note here that even if the effective advection is ignored in the StSm model, the two models still differ in a general case due to the first two terms in (13).

Smagorinsky's pioneering work remains to date a popular model for LES, however it has certain associated drawbacks. The model assumes the existence of equilibrium between the kinetic energy flux across scale and the large scales of turbulence - this equilibrium is not established in many cases such as the leading edge of an airplane wing or turbulence with strong buoyancy. In addition, a form of arbitrariness and user dependency is introduced due to the presence of the Smagorinsky coefficient. This coefficient is assumed to be constant irrespective of position and time. Lilly (38) suggests a constant

coefficient value to be appropriate for the Smagorinsky model. However this was disproved by the works of (12), and (39) who shows that a constant value did not efficiently capture turbulence especially in boundary layers.

Numerous attempts were made to correct for the fixed constant such as damping functions (40) or renormalisation group theory (41). Germano *et al.* (39) provided a non *ad-hoc* manner of calculating the Smagorinsky coefficient varying with space and time using the Germano identity and an additional test filter t (termed as the Dynamic Smagorinsky (DSmag) model).

$$\mathcal{L}_{ij} = T_{ij} - \tau_{ij}^t, \quad (14)$$

where τ stands for the SGS stress filtered by the test filter t , T is the filtered SGS stress calculated from the test filtered velocity field, and $\mathcal{L}$ stands for the resolved turbulent stress. The Smagorinsky coefficient can thus be calculated as:

$$C_s^2 = \frac{\langle \mathcal{L}_{ij} M_{ij} \rangle}{\langle M_{ij} M_{ij} \rangle}, \text{ where} \quad (15)$$

$$M_{ij} = -2\Delta^2(\alpha'^2 \|\bar{\mathbf{S}}^t\| \bar{S}_{ij}^t - (\|\bar{\mathbf{S}}\| \bar{S}_{ij})^t) \quad (16)$$

and α' stands for the ratio between the test filter and the LES filter.

The dynamical update procedure removes the user dependancy aspect in the model, however it introduces unphysical values for the coefficient at certain instances. An averaging procedure along a homogenous direction is necessary to provide physical values for C_s . However, most turbulent flows, foremost being wake flow around a cylinder, lack a homogenous direction for averaging. In such cases, defining the coefficient is difficult and needs ad-hoc measures such as local averaging, threshold limitation, and/or filtering methods to provide nominal values for C_s . For the present study, we focus on the classical and dynamic variations of the Smagorinsky model - these model were used to study cylinder wake flow by (8), (42), (43), among many others.

2.0.2. Local variance based models:

As the name states, the variance tensor can be calculated by an empirical covariance of the resolved velocity within a specified local neighbourhood. The neighbourhood can be spatially or temporally located giving rise to two formulations. A spatial neighbourhood based calculation (referred to as

Stochastic Spatial Variance model (StSp)) is given as:

$$\mathbf{a}(\mathbf{x}, n\delta t) = \frac{1}{|\Gamma| - 1} \sum_{x_i \in \eta(x)} (\mathbf{u}(x_i, n\delta t) - \bar{\mathbf{u}}(x, n\delta t))(\mathbf{u}(x_i, n\delta t) - \bar{\mathbf{u}}(x, n\delta t))^T C_{sp}, \quad (17)$$

where $\bar{u}(x, n\delta t)$ stands for the empirical mean around the arbitrarily selected local neighbourhood defined by Γ . The constant C_{sp} is defined as (18):

$$C_{sp} = \left(\frac{\ell_{res}}{\eta} \right)^{\frac{5}{3}} \Delta t, \quad (18)$$

where ℓ_{res} is the resolved length scale, η is the Kolmogorov length scale and Δt is the simulation time step.

A similar local variance based model can be envisaged in the temporal framework; however, it has not been analysed in this paper due to memory limitations.

It is important to note that the prefix stochastic has been added to the Mulu to differentiate the Mulu version of the Smagorinsky model from its classical purely deterministic version. The model equations while derived using stochastic principles are applied in this work in a purely deterministic sense. The full stochastic formulation of Mulu has been studied by (17).

3. Flow configuration and numerical methods

The flow was simulated using a parallelised flow solver, Incompact3d, developed by (44). Incompact3d relies on a sixth order finite difference scheme (the discrete schemes are described in (45)) and the Immersed Boundary Method (IBM) (for more details on IBM refer to (46)) to emulate a body forcing. The main advantage of using IBM is the ability to represent the mesh in cartesian coordinates and the straightforward implementation of high-order finite difference schemes in this coordinate system. The IBM in Incompact3d has been applied effectively to cylinder wake flow by (7) and to other flows by (46), and (47) among others. A detailed explanation of the IBM as applied in Incompact3d, as well as its application to cylinder wake flow can also be found in (7). It is important to note that this paper focuses on the accuracy of the sub-grid models within the code and not on the numerical methodology (IBM/numerical schemes) of the code itself.

The incompressibility condition is treated with a fractional step method based on the resolution of a Poisson equation in spectral space on a staggered

pressure grid combined with IBM. While solving the Poisson equation for the stochastic formulation, the velocity bias was taken into account in order to satisfy the stochastic mass conservation constraints. It can be noted that although the solution of the Poisson equation in physical space is computationally heavy, the same when performed in Fourier space is cheap and easily implemented with Fast Fourier transforms. For more details on Incompact3d the authors refer you to (44) and (48).

The flow over the cylinder is simulated for a Re of 3900 on a domain measuring $20D \times 20D \times \pi D$. The cylinder is placed in the centre of the lateral domain at $10D$ and at $5D$ from the domain inlet. For statistical purposes, the centre of the cylinder is assumed to be $(0, 0)$. A coarse mesh resolution of $241 \times 241 \times 48$ is used for the coarse LES (cLES). cLES discretisation has been termed as coarse as this resolution is $\sim 6.2\%$ the resolution of the reference LES of (7) (henceforth referred to as LES - Parn). In terms of Kolmogorov units (η), the mesh size for the cLES is $41\eta \times 7\eta - 60\eta \times 32\eta$. The Kolmogorov length scale has been calculated based on the dissipation rate and viscosity, where the dissipation rate can be estimated as $\epsilon \sim U^3/L$ where U and L are the characteristic velocity scale and the integral length scale. A size range for y is used due to mesh stretching along the lateral (y) direction which provides a finer mesh in the middle. Despite the stretching, the minimum mesh size for the cLES is still larger than the mesh size of particle imagery velocimetry (PIV) reference measurements of (7) (henceforth referred to as PIV - Parn). For all simulations, inflow/outflow boundary condition is implemented along the streamwise (x) direction with free-slip and periodic boundary conditions along the lateral (y) and spanwise (z) directions respectively - the size of the spanwise domain has been fixed to πD as set by (8), which was also validated by (7) to be sufficient with periodic boundary conditions. The turbulence is initiated in the flow by introducing a white noise in the initial condition. Time advancement is performed using the third order Adam-Bashforth scheme. A fixed coefficient of 0.1 is used for the Smagorinsky models as suggested in literature (43) while a spatial neighbourhood of $7 \times 7 \times 7$ is used for the Stochastic Spatial model. For the dynamic Smagorinsky model, despite the lack of clear homogenous direction, a spanwise averaging is employed. In addition, the constant is filtered and a threshold on negative and large positive coefficients is also applied to stabilise the model. Note that the positive threshold is mesh dependant and needs user-intervention to specify the limits.

The reference PIV (7) was performed with a cylinder of diameter 12 mm

and 280 mm in length placed $3.5D$ from the entrance of the testing zone in a wind tunnel of length 100 cm and height 28 cm. Thin rectangular end plates placed 240 mm apart were used with a clearance of 20 mm between the plates and the wall. 2D2C measurements were carried out at a free stream velocity of 4.6 m s^{-1} ($Re \sim 3900$) recording 5000 image pairs separated by $25 \mu\text{s}$ with a final interrogation window measuring 16×16 pixels with relatively weak noise. For more details about the experiment refer to (7).

The high resolution LES of (7) was performed on Incompact3d on the same domain measuring $20D \times 20D \times \pi D$ with $961 \times 961 \times 48$ cartesian mesh points. The simulation was performed with the structure function model of (49) with a constant mesh size. LES - Parn is well resolved, however, there is a distinct statistical mismatch between LES - Parn and PIV-Parn especially along the centre-line (see figure 1a and figure 1b). Literature suggests that the wake behind the cylinder at a $Re \sim 3900$ is highly volatile and different studies predict slightly varied profiles for the streamwise velocity along the centre-line. The averaging time period, the type of model, and the mesh type all affect the centre-line velocity profile. As can be seen in figure 1a and figure 1b, each reference data set predicts a different profile/magnitude for the streamwise velocity profiles. The DNS study of (6) does not present the centreline velocity profiles. This provided the motivation for performing a DNS study at $Re \sim 3900$ to accurately quantify the velocity profiles and to reduce the mismatch between the existing experimental and simulation datasets. The DNS was performed on the same domain with $1537 \times 1025 \times 96$ cartesian mesh points using Incompact3d with stretching implemented in the lateral (y) direction.

From figure 1a we can see that the DNS and the PIV of Parnaudeau are the closest match among the data sets while significant deviation is seen in the other statistics. In the fluctuating streamwise velocity profiles, the only other data sets that exist are of (50) who performed Laser Doppler Velocimetry (LDV) experiments at $Re = 3000$ and $Re = 5000$. Among the remaining data-sets (LES of Parnaudeau, PIV of Parnaudeau, and current DNS) matching profiles are observed for the DNS and PIV despite a magnitude difference. These curves also match the profiles obtained by the experiments of Norberg (50) in shape, i.e. the similar magnitude, dual peak nature. The LES of Parnaudeau (7) is the only data-set to estimate an inflection point and hence is not considered further as a reference. The lower energy profile of the PIV may be attributed to the methods used for calculating the vector fields which employ a large-scale representation of the flow via interrogation win-

Table 1: Flow parameters.

	Re	$n_x \times n_y \times n_z$	$l_x/D \times l_y/D \times l_z/D$	$\Delta x/D$	$\Delta y/D$	$\Delta z/D$	$U\Delta t/D$
cLES	3900	$241 \times 241 \times 48$	$20 \times 20 \times \pi$	0.083	0.024-0.289	0.065	0.003
DNS	3900	$1537 \times 1025 \times 96$	$20 \times 20 \times \pi$	0.013	0.0056-0.068	0.033	0.00075
PIV - Parn	3900	$160 \times 128 \times 1$	$3.6 \times 2.9 \times 0.083$	0.023	0.023	0.083	0.01
LES - Parn	3900	$961 \times 961 \times 48$	$20 \times 20 \times \pi$	0.021	0.021	0.065	0.003

dows similar to a LES resolution (51). The DNS, however, exhibits a profile similar to other references and a magnitude in between the two LDV experiments of Norberg. Considering the intermediate Reynolds number of the DNS compared to the Norberg experiments, this suggests good convergence and accuracy of the DNS statistics. Note that the cLES mesh is $\sim 0.46\%$ the cost of the DNS. Table 1 concisely depicts all the important parameters for the flow configuration as well as the reference datasets.

Wake flow around a cylinder was simulated in the above enumerated configuration with the following SGS models: Classic Smagorinsky (Smag), Dynamic Smagorinsky (DSmag), Stochastic Smagorinsky (StSm), and Stochastic Spatial (StSp) variance. In accordance with the statistical comparison performed by (8), first and second order temporal statistics have been compared at 3 locations ($x = 1.06D$ (top), $x = 1.54D$ (middle), and $x = 2.02D$ (bottom)) in the wake of the cylinder. All cLES statistics are computed (after an initial convergence period) over 90,000 time steps corresponding to 270 non-dimensional time or ~ 54 vortex shedding cycles. All statistics are also averaged along the spanwise (z) direction. The model statistics are evaluated against the PIV experimental data of (7) and the DNS for which the data has been averaged over 400,000 time steps corresponding to 60 vortex sheddings. The work of (7) suggests that at least 52 vortex sheddings are needed for convergence which is satisfied for all the simulations. In addition, spanwise averaging of the statistics results in converged statistics comparable with the PIV ones. Both DNS and PIV statistics are provided for all statistical comparison, however, the DNS is used as the principal reference when an ambiguity exists between the two references.

4. Results

In this section, we present the model results, performance analysis and physical interpretations. Firstly, The cLES results are compared with the

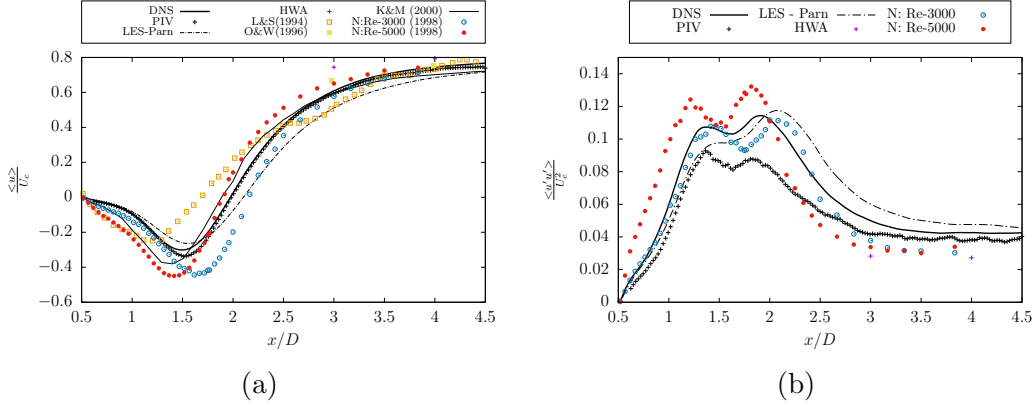


Figure 1: Mean streamwise velocity (a) and fluctuating streamwise velocity (b) in the streamwise direction along the centreline ($y = 0$) behind the cylinder for the reference data-sets. Legend: HWA - hot wire anemometry(7), K&M - B-spline simulations (case II) of Kravchenko and Moin(4), L&S - experiment of Lourenco and Shih (9), N - experiment of Norberg at Re = 3000 and 5000 (50), O&W - experiement of Ong and Wallace (52)

reference PIV and the DNS. The focus on centreline results for certain comparisons is to avoid redundancy and because these curves show maximum statistical deviation. This is followed by a characterisation and physical analysis of the velocity bias and SGS contributions for the MULU. The section is concluded with the computation costs of the different models.

4.1. Coarse LES

For cLES, the MULU have been compared with classic and dynamic version of the Smagorinsky model, DNS, and PIV - Parn. Figure 2 and figure 3 depict the mean streamwise and lateral velocity respectively plotted along the lateral (y) direction. In the mean streamwise velocity profile (see figure 2a), the velocity deficit behind the cylinder depicted via the U-shaped profile in the mean streamwise velocity is captured by all models. The expected downstream transition from U-shaped to a V-shaped profile is seen for all the models - a delay in transition is observed for Smag model which biases the statistics at $x = 1.54D$ and $2.02D$. For the mean lateral component (see figure 3), all models display the anti-symmetric quality with respect to $y = 0$. Smag model shows maximum deviation from the reference DNS statistics in all observed profiles. All models but Smag capture the profile well while broadly StSp and DSmag models better capture the magnitude. As a general trend, Smag model can be seen to under-predict statistics while StSm model

over-predicts.

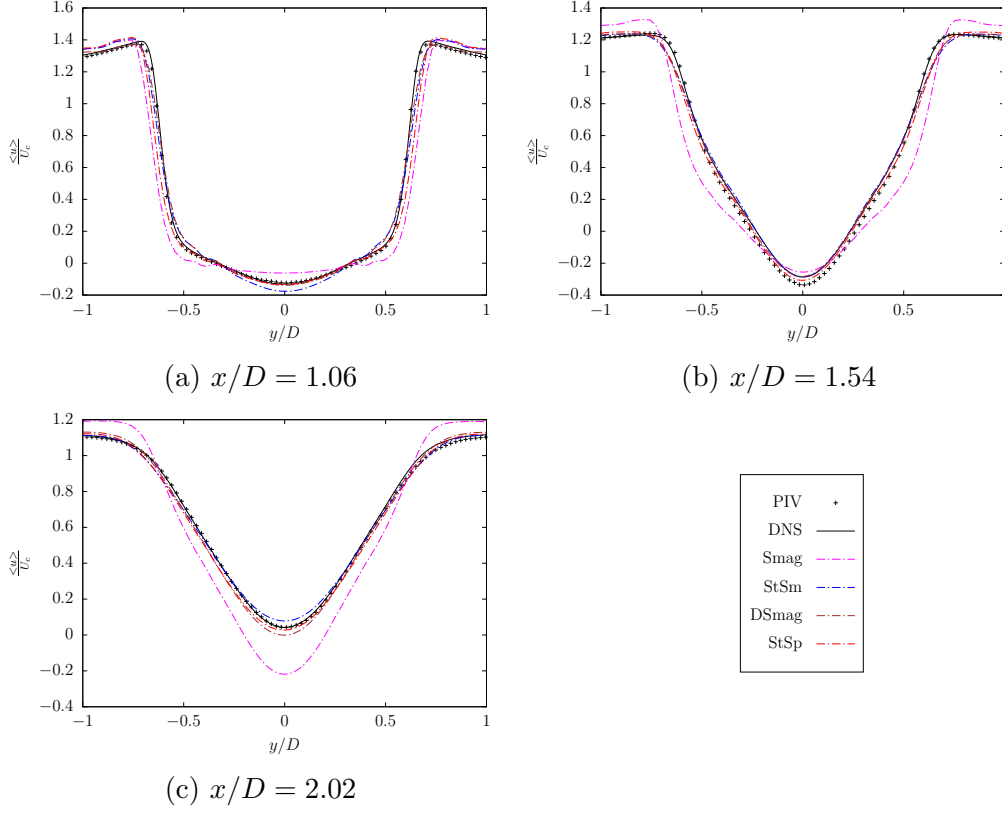


Figure 2: Mean streamwise velocity at 1.06D (top), 1.54D (middle), and 2.02D (bottom) in the wake of the circular cylinder.

A better understanding of the model performance can be obtained through Figure 4 - 6 which depict the second order statistics, i.e. the rms component of the streamwise ($\langle u'u' \rangle$) and lateral ($\langle v'v' \rangle$) velocity fluctuations and the cross-component ($\langle u'v' \rangle$) fluctuations. The transitional state of the shear layer can be seen in the reference statistics by the two strong peaks at $x = 1.06D$ in figure 4a. The magnitude of these peaks is in general under-predicted, however, a best estimate is given the MULU. DSmag and Smag models can be seen to under-predict these peaks at all x/D . This peak is eclipsed by a stronger peak further downstream due to the formation of the primary vortices (see figure 4b) which is captured by all the models.

The maxima at the centreline for figure 5 and the anti-symmetric structure for figure 6 are seen for all models. Significant mismatch is observed

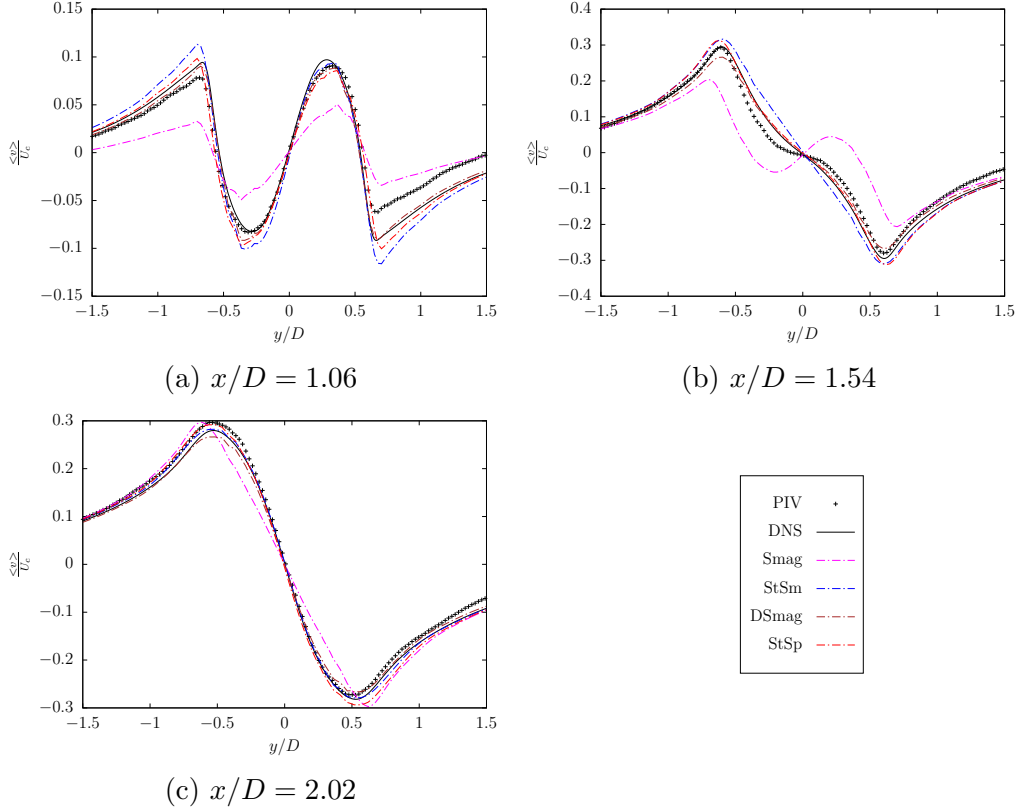


Figure 3: Mean lateral velocity at 1.06D, 1.54D, and 2.02D in the wake of the circular cylinder.

between the reference and the Smag/StSm models especially in figure 5a and 6a. In all second-order statistics, StSm model improves in estimation as we move further downstream. No such trend is seen for StSp or DSmag models while a constant under-prediction is seen for all Smag model statistics. This under-prediction could be due to the inherent over-dissipativeness of the Smagorinsky model which smooths the velocity field. This is corrected by DSmag/StSm models and in some instances over-corrected by the StSm model. A more detailed analysis of the two formulations under location uncertainty (StSm and StSp) is presented in sections 4.2.

The smoothing for each model is better observed in the 3D isocontours of vorticity modulus (Ω) plotted in figure 7. Plotted at non-dimensional $\Omega = 7$, the isocontours provide an understanding of the dominant vortex structures within the flow. While large-scale vortex structures are observed in all flows,

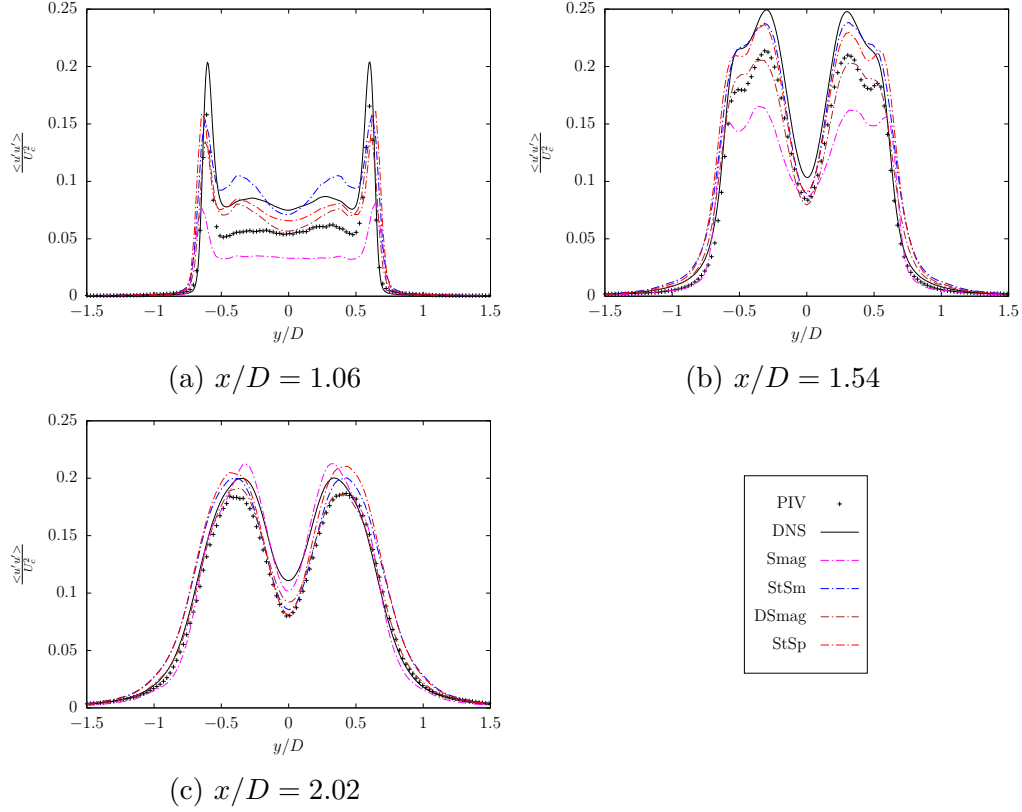


Figure 4: Streamwise rms velocity ($u'u'$) fluctuations at 1.06D, 1.54D, and 2.02D in the wake of the circular cylinder.

the small-scale structures and their spatial extent seen in the DNS are better represented by the MULU. The over-dissipativeness of the Smag model leads to smoothed isocontours with reduced spatial extent. The large-scale vortex structures behind the cylinder exhibit the spanwise periodicity observed by Williamson (53) for cylinder wake flow at low $Re \sim 270$. Inferred to be due to mode B instability by Williamson, this spanwise periodicity was associated with the formation of small-scale streamwise vortex pairs. It is interesting to observe here the presence of similar periodicity at higher Re - this periodicity will be further studied at a later stage in this paper.

A stable shear layer associated with higher dissipation is observed in Smag model with the shear layer instabilities beginning further downstream than the MULU. An accurate shear layer comparison can be done by calculating the recirculation length (L_r) behind the cylinder. Also called the

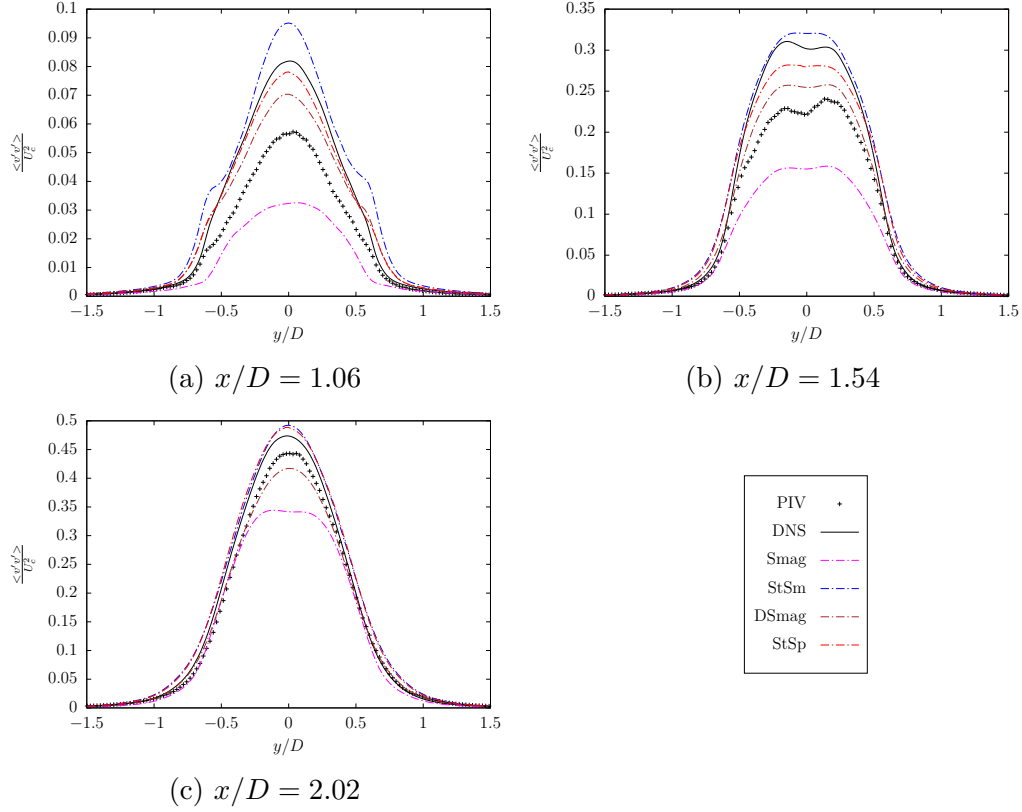


Figure 5: Lateral rms velocity ($v'v'$) fluctuations at 1.06D, 1.54D, and 2.02D in the wake of the circular cylinder.

bubble length, it is the distance between the base of the cylinder and the point with null longitudinal mean velocity on the centreline of the wake flow. This parameter has been exclusively studied due to its strong dependence on external disturbances in experiments and numerical methods in simulations (54; 4). The effective capture of the recirculation length leads to the formation of U-shaped velocity profile in the near wake while the presence of external disturbances can lead to a V-shaped profile as obtained by the experiments of (9). Parnaudeau *et al.* (7) used this characteristic to effectively parameterise their simulations.

The instantaneous contours can provide a qualitative outlook on the recirculation length based on shear layer breakdown and vortex formation. However, in order to quantify accurately the parameter, the mean and rms streamwise velocity fluctuation components were plotted in the streamwise

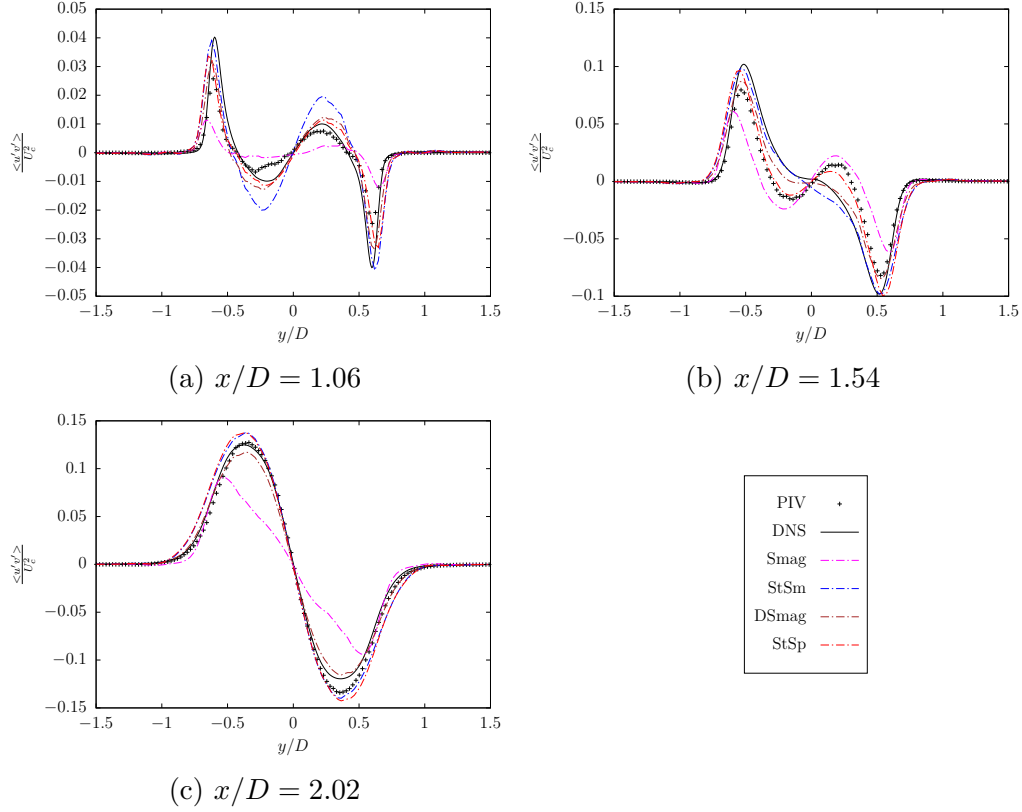
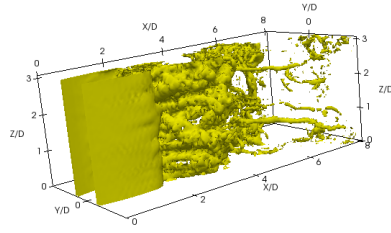


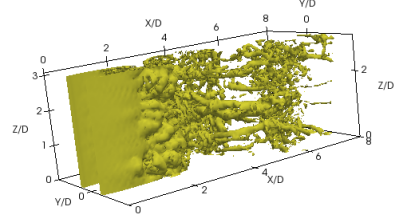
Figure 6: Rms velocity fluctuations cross-component ($u'v'$) at 1.06D, 1.54D, and 2.02D in the wake of the circular cylinder.

(x) direction along the centreline (see figure 8a, and figure 8b). The recirculation length for each model is tabulated in table 2. StSp and DSmag models capture the size of the recirculation region with 0% error while the StSm model under estimates the length by 5.9% and the Smag model over estimates by 15.9%. The magnitude at the point of inflection is accurately captured by all the models (figure 8a).

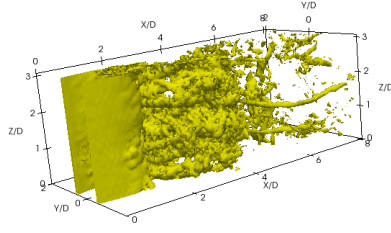
For the rms centreline statistics of figure 8b, due to ambiguity between references, the DNS is chosen for comparison purposes. However, the similar magnitude, dual peak nature of the profile can be established through both the references. This dual peak nature of the model was also observed in the experiments of (50) who concluded that within experimental accuracy, the secondary peak was the slightly larger RMS peak as seen for the DNS. The presence of the secondary peak is attributed to the cross over of mode B



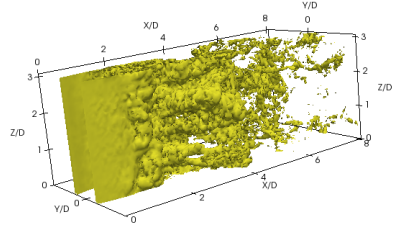
(a) Smag,



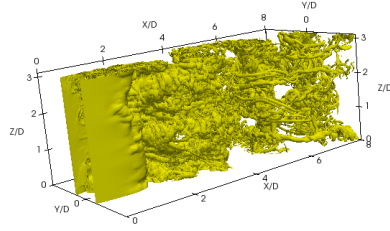
(b) DSmag,



(c) StSm.



(d) StSp,



(e) DNS,

Figure 7: 3D instantaneous vorticity iso-surface at $\Omega = 7$.

secondary structures within a Re regime (in the transitional regime) of regular shedding frequency with minimal undulations of the Kármán vortices along the spanwise direction. At higher Re above the transition range ($\text{Re} > 5 \times 10^3$), where the shedding frequency is irregular accompanied by significant spanwise undulations of the streamwise vortices, the primary peak is smoothed into an inflection point. This is similar to the profile obtained

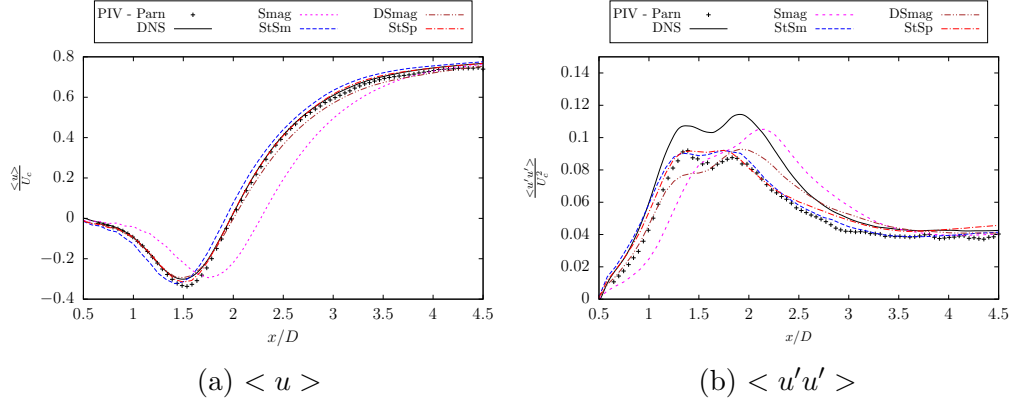


Figure 8: Mean (a) and Fluctuating (b) streamwise velocity profile in the streamwise direction along the centreline behind the cylinder.

for LES-Parn in figure 1b despite the simulation being within the transition regime.

The fluctuating centreline velocity profiles for the deterministic Smagorinsky models display an inflection point unlike the references. The Mulu display a hint of the correct dual peak nature while under-predicting the magnitude matching with the PIV's large scale magnitude rather than the DNS. Although the Smag model has a second peak magnitude closer to the DNS, the position of this peak is shifted farther downstream. This combined with the inability of the model to capture the dual-peak nature speaks strongly against the validity of the Smag model statistics. Further analysis can be done by plotting 2D isocontours of the streamwise fluctuating velocity behind the cylinder, as shown in figure 9. The isocontours are averaged in time and along the spanwise direction. The profiles show a clear distinction between the classical models and the Mulu in the vortex bubbles just behind the recirculation region. The vortex bubbles refer to the region in the wake where the initial fold-up of the vortices start to occur from the shear layers. The Mulu match better with the DNS isocontours within this bubble as compared to the Smag or DSmag models. Along the centreline, Mulu under-predict the magnitude, as depicted by the lower magnitude dual peaks in figure 8b. As we deviate from the centre-line, the match between the Mulu and the DNS improves considerably. The mismatch of the isocontours in the vortex bubbles for the Smag and DSmag models with the DNS suggests that a higher magnitude for the centreline profile is not indicative

of an accurate model.

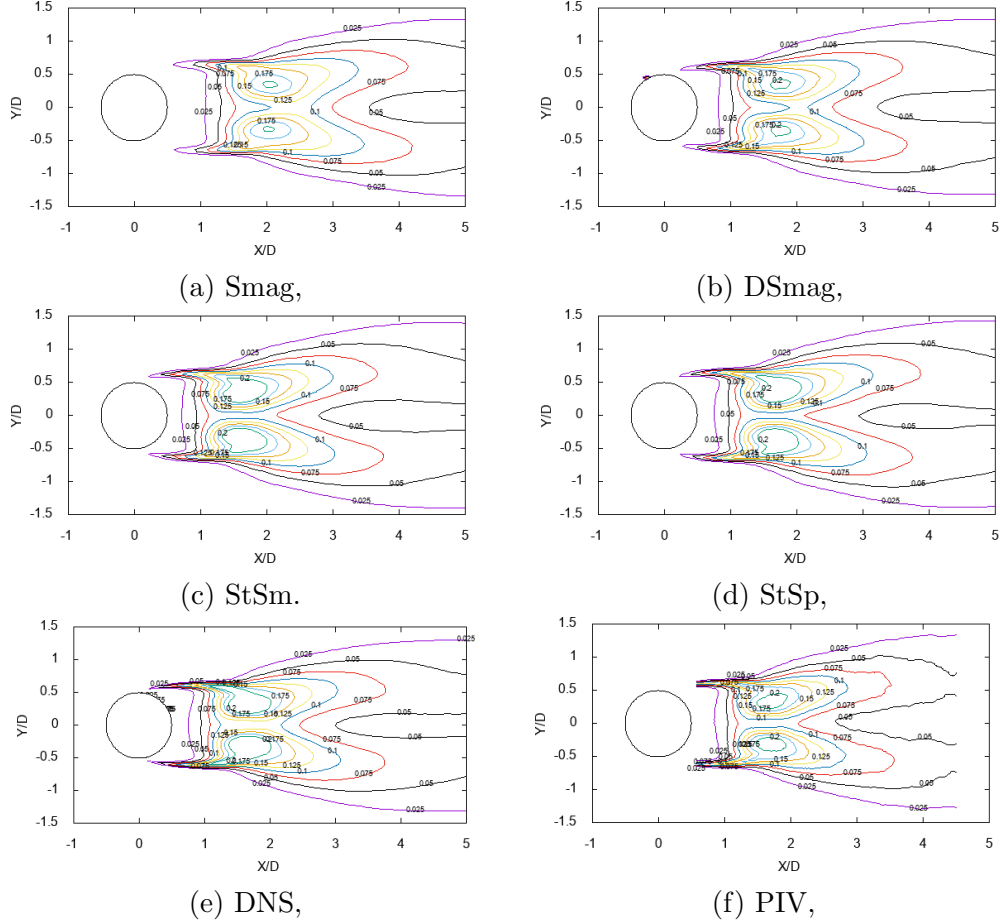


Figure 9: 2D isocontours of time averaged fluctuating streamwise velocity ($u'u'$).

The dual peak nature of the streamwise velocity rms statistics show a strong dependance on the numerical model and parameters. This can be better understood via the constant definition within the StSp model formulation (refer to (18)). The constant requires definition of the scale (ℓ_{res}) of the simulation which is similar to Δ used in classic Smagorinsky model, i.e. it defines the resolved length scale of the simulation. In the case of stretched mesh, the definition of this ℓ_{res} can be tricky due to the lack of a fixed mesh size. It can be defined as a maximum ($\max(dx, dy, dz)$) or a minimum ($\min(dx, dy, dz)$) or an average $(dx * dy * dz)^{(1/3)}$. A larger value

Table 2: Recirculation lengths for cLES.

Model	PIV - Parn	DNS	Smag	DSmag	StSm	StSp
L_r/D	1.51	1.50	1.75	1.50	1.42	1.50

of this parameter would signify a coarser mesh (i.e. a rough resolution) while a small value indicates a finer cut off scale or a finer mesh resolution. When ℓ_{res} is large, corresponding to a “PIV resolution”, the centreline streamwise rms statistics display a dual peak nature with a larger initial peak similar to PIV reference. A smaller value for ℓ_{res} , corresponding to a “higher resolution LES”, shifts this dual peak into a small initial peak and a larger second peak similar to the DNS and of higher magnitude. The statistics shown above have been obtained with an ℓ_{res} defined as $(\max(dx, dy, dz))$ to emulate the coarseness within the model.

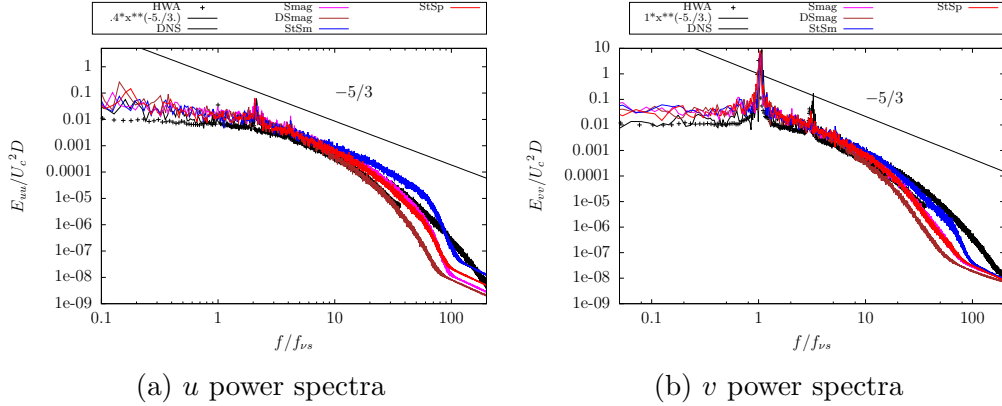


Figure 10: Power spectra of streamwise (a) and lateral (b) velocity component at $x/D = 3$ behind the cylinder.

Figure 10a and figure 10b show the power spectra of the streamwise and lateral velocity fluctuation calculated over time using probes at $x/D = 3D$ behind the cylinder along the full spanwise domain. For the model power spectra, 135,000 time-steps were considered corresponding to a non-dimensional time of 405 which encapsulates ~ 81 vortex shedding cycles. The hanning methodology is used to calculate the power spectra with an overlap of 50% considering 30 vortex shedding cycles in each sequence. The

reference energy spectra (namely HWA) have been obtained from (7) while the DNS energy spectra has been calculated similar to the cLES. The process of spectra calculation for both reference and models are identical. All the values have been non-dimensionalised.

The fundamental harmonic frequency ($f/f_s = 1$) and the second harmonic frequency are captured accurately by all models in the v-spectra. Let us recall that the cLES mesh is coarser than the PIV grid. Twice the vortex shedding frequency is captured by the peak in u-spectra at $f/f_s \sim 2$ as expected - twice the Strouhal frequency is observed due to symmetry condition at the centreline (6). The HWA measurement has an erroneous peak at $f/f_s \sim 1$ which was attributed to calibration issues and the cosine law by (7). All models match with both the reference spectra. One can observe a clear inertial subrange for all models in line with the expected $-5/3$ slope. The models in the order of increasing energy at the small scales is DSmag < Smag = StSp < StSm. For the StSm model, an accumulation of energy is observed at the smaller scales in the u-spectra, unlike the StSp model. This suggests that the small-scale fluctuations seen in vorticity or velocity contours for the StSp model (i.e. in figure 7) are physical structures and not a numerical accumulation of energy at the smaller scales known to occur for LES.

The statistical comparisons show the accuracy and applicability of the MULU. The next sub-section focuses on the physical characterisation of the MULU - SGS dissipation, velocity bias and their contributions are studied in detail.

4.2. Velocity bias characterisation

The functioning of the MULU is through the small-scale velocity auto-correlation **a**. The effect of this parameter on the simulation is threefold: firstly, it contributes to a velocity bias/correction which is a unique feature of the MULU. Secondly, this velocity correction plays a vital part in the pressure calculation to maintain incompressibility. Finally, it contributes to the SGS dissipation similar to classical LES models - this signifies the dissipation occurring at small scales. This threefold feature of the MULU is explored in this section.

The contribution of the velocity bias can be characterised by simulating the MULU (StSm and StSp) with and without the velocity bias (denoted by Nad for no advection bias) and comparing the statistics (see figure 11a - 11b). Only the centre-line statistics have been shown for this purpose as

they display maximum statistical variation among the models and provide an appropriate medium for comparison. For the simulations without the velocity bias, the convective part in the NS equations remains purely a function of the large-scale velocity. In addition, the weak incompressibility constraint (5) is not enforced in the simulations with no velocity bias and the pressure is computed only on the basis of large-scale velocity. Similar to the Smagorinsky model, where the gradients of the trace of the stress tensor are considered subsumed within the pressure term, the divergence of the velocity bias is considered subsumed within the pressure term. The simulations parameters and flow configuration remain identical to cLES configuration.

The statistics show improvement in statistical correlation when the velocity bias is included in the simulation - all statistical profiles show improvement with inclusion of velocity bias but only the centre-line statistics have been shown to avoid redundancy. In the mean profile, the inclusion of velocity bias appears to correct the statistics for both models to match better with the reference. For the StSm model, there is a right shift in the statistics while the opposite is seen for the StSp model. The correction for the StSp model appears stronger than that for the StSm model. This is further supported by the fluctuation profile where without the velocity bias, the StSp model tends to the Smag model with an inflection point while the inclusion of a connection between the large-scale velocity advection and the small-scale variance results in the correct dual peak nature of the references. For the StSm model, figure 11b suggests a reduction in statistical correlation with the inclusion of the velocity bias - this is studied further through 2D isocontours.

Figure 12 plots 2D isocontours for the streamwise fluctuating profiles for the MULU. Once again an averaging is performed in time and along the spanwise direction. A clear distinction between the models with and without velocity bias is again difficult to observe. However, on closer inspection, within the vortex bubbles, we can see that including velocity bias improves the agreement with the DNS by reducing the bubble size for the StSm model and increasing it for the StSp model. The higher magnitude prediction along the centreline seen for the StSm - Nad model could be the result of an overall bias of the statistics and not due to an improvement in model performance - the presence of an inflection point in the profile further validates the model inaccuracy. This error is corrected in the model with velocity bias. This corrective nature of the bias is further analysed.

For the StSp model, the simulation without the bias has a larger recir-

culation zone or is "over dissipative" and this is corrected by the bias. This result supports the findings of (55) whose structure model, when employed in physical space, applies a similar statistical averaging procedure of square-velocity differences in a local neighbourhood. They found their model to also be over-dissipative in free-shear flows and did not work for wall flows as too much dissipation suppressed development of turbulence and had to be turned off in regions of low three-dimensionality. To achieve that (56) proposed the filtered-structure-function model, which removes the large-scale fluctuations before computing the statistical average. They applied this model with success to large-eddy simulation and analysis of transition to turbulence in a boundary layer. For the StSp model, which also displays this over dissipation quality (without velocity bias), the correction appears to be implicitly done by the velocity bias. Such a velocity correction is consistent with the recent findings of (19) who provided physical interpretations of the local corrective advection due to the turbulence inhomogeneity in the pivotal region of the near wake shear layers where transition to turbulence takes place. The recirculation length for all cases is tabulated in table 3. Data are obtained from the centre-line velocity statistics shown in figure 11a. The tabulated values further exemplify the corrective nature of the velocity bias where we see an improved estimation of the recirculation length with the inclusion of the velocity bias. Also, a marginal improvement in statistical match similar to figure 11a is observed with the inclusion of the velocity bias for all lateral profiles (not shown here). It can be concluded that the inclusion of velocity bias provides, in general, an improvement to the model.

The physical characteristics of the velocity bias (expressed henceforth as $\mathbf{u}^* = \frac{1}{2}\nabla \cdot \mathbf{a}$) are explored further. The bias $\mathbf{u}^*$, having the same units as velocity, can be seen as an extension or a correction to velocity. Extending this analogy, the divergence of $\mathbf{u}^*$ is similar to "the divergence of a velocity field". This is the case in the MULK where to ensure incompressibility, the divergence free constraints (eq. (5)) are necessary. The stability and statistical accuracy of simulations were improved with a pressure field calculated using the modified velocity $\tilde{\mathbf{u}}$, i.e. when the weak incompressibility constraint was enforced on the flow. This pressure field can be visualised as a true pressure field unlike the Smagorinsky model where the gradients of the trace of the stress tensor are absorbed in an effective pressure field.

Stretching the $\mathbf{u}^*$ and velocity analogy, we can also interpret the curl of $\mathbf{u}^*$ ($\nabla \times \mathbf{u}^*$) as vorticity or more specifically as a vorticity bias. The curl of $\mathbf{u}^*$ plays a role in the wake of the flow where it can be seen as a correction

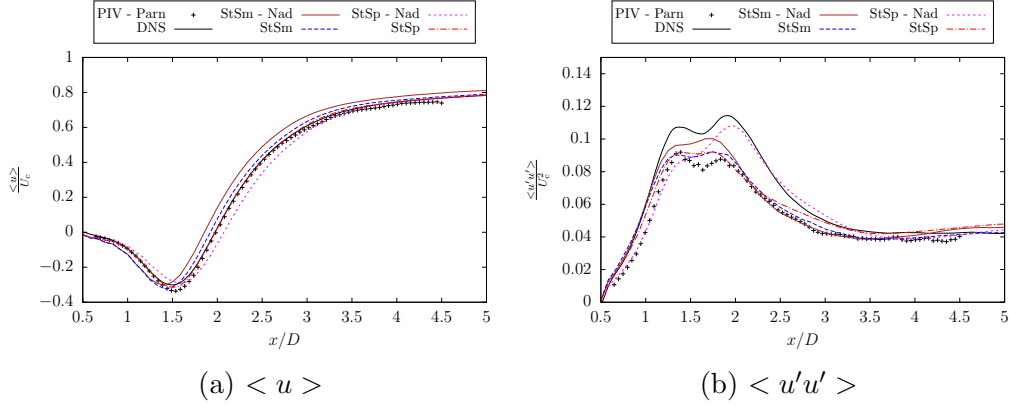


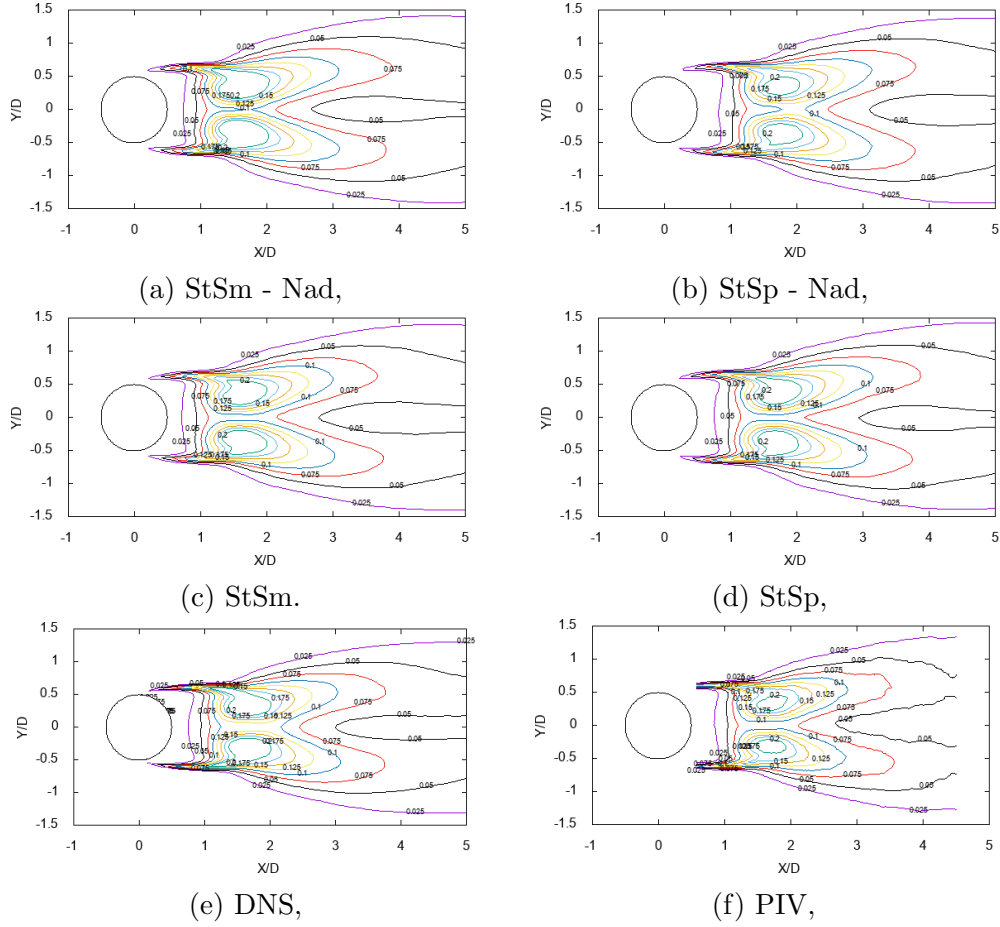
Figure 11: Effect of velocity bias on centre-line mean (a) and fluctuating (b) streamwise velocity behind the cylinder.

Table 3: Recirculation lengths with and without velocity bias.

Model	PIV - Parn	DNS	StSm - Nad	StSm	StSp - Nad	StSp
L_r/D	1.51	1.50	1.42	1.42	1.58	1.50

to the vorticity field. The divergence and curl of $\mathbf{u}^*$ are features solely of the MULU and their characterisation defines the functioning of these models.

Figure 13 depicts the mean isocontour of $\nabla \cdot (\mathbf{u}^*) = 0.02$ for the two MULU. This divergence function is included in the poisson equation for pressure calculation in order to enforce the weak incompressibility constraint. In the StSm model the contribution is strictly limited to within the shear layer while in the StSp model the spatial influence extends far into the downstream wake. The stark difference in the spatial range could be due to the lack of directional dissipation in the StSm model which is modelled on the classic Smagorinsky model. This modelling results in a constant diagonal auto correlation matrix, and the trace elements simplifying to a laplacian of $\mathbf{a}$ ($\Delta \mathbf{a}$) for $\nabla \cdot (\mathbf{u}^*)$. This formulation contains a no cross-correlation assumption (zero non-diagonal elements in the auto correlation matrix) as well as ignoring directional dissipation contribution (constant diagonal terms provides equal SGS dissipation in all three principle directions). These Smagorinsky like assumptions place a restriction on the form and magnitude of $\mathbf{u}^*$ which are absent in the StSp model. The existence of cross-correlation terms in



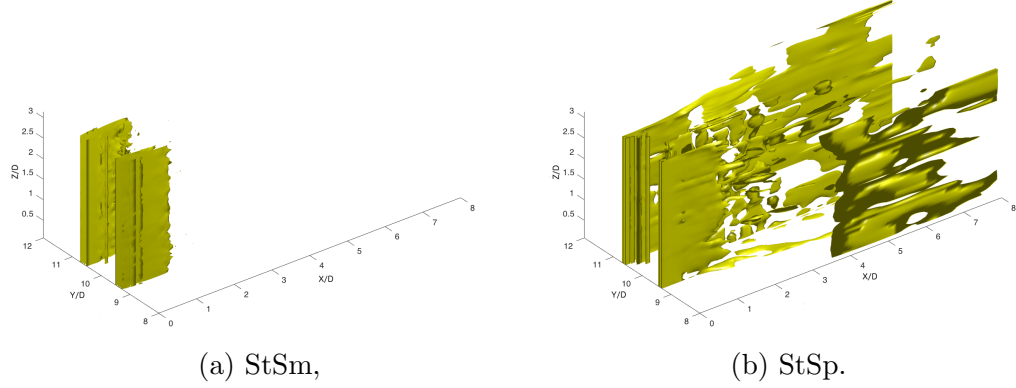


Figure 13: 3D isocontour of the divergence of $\mathbf{u}^*$ at $\nabla \cdot (\mathbf{u}^*) = 0.02$

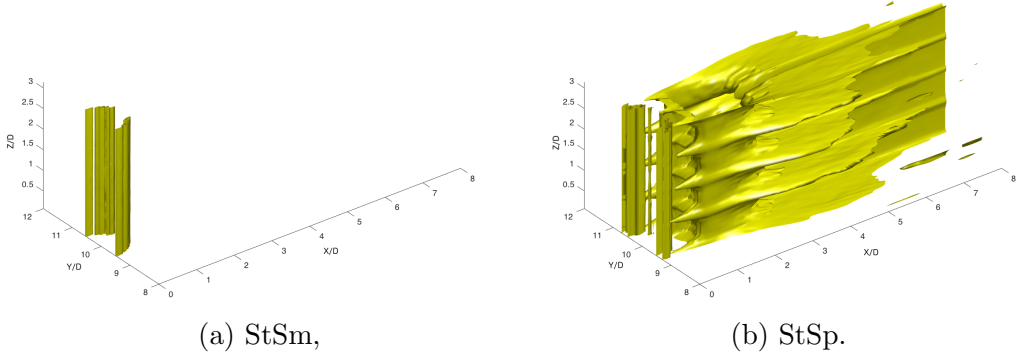
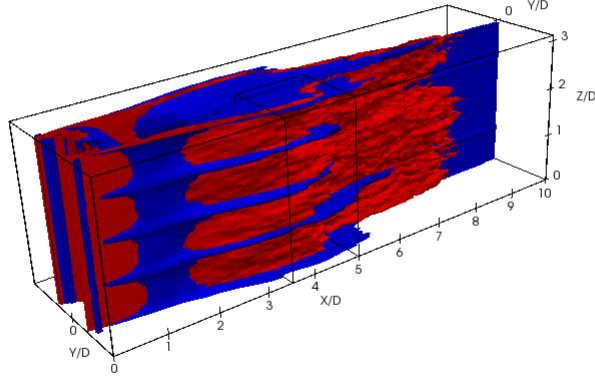


Figure 14: 3D isocontour of the curl of $\mathbf{u}^*$ at $\nabla \times (\mathbf{u}^*) = 0.01$ (for StSm), 0.05 (for StSp)

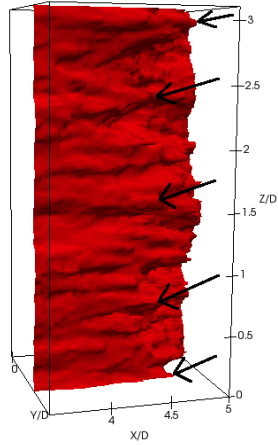
the isocontour of the curl of $\mathbf{u}^*$ for the StSp model at $\nabla \times (\mathbf{u}^*) = 0.05$ with the mean DNS flow vorticity isocontour at $\Omega = 3$. Figure 15a shows the superposition over a large spatial domain behind the cylinder. The outlined box within figure 15a indicates the area of zoom for figure 15b and figure 15c. Figure 15b is the zoomed in view of the mean DNS vorticity isocontour at $\Omega = 3$. The arrows in figure 15b indicate the points of match between the mean vorticity peaks and the peaks in the mean curl isocontour of $\mathbf{u}^*$ for the StSp model. The superposed isocontours are shown in figure 15b. The structural match between the peaks of the two isocontours are satisfactory with the periodic peaks in the curl of $\mathbf{u}^*$ matching that of the noisier vorticity

isocontour. While clear periodicity is not observed for the mean vorticity, alternate peaks and troughs can be seen which match with the peaks in the mean curl isocontour. The wavelength of this periodicity is comparable with the spanwise wavelength of approximately 1D of mode B instabilities observed by (53) for $Re \sim 270$. The footprint of mode B instabilities is linked to secondary instabilities leading to streamwise vortices observed for Re ranging from 270 to ~ 21000 (57). These results demonstrate the ability of the spatial variance model to capture the essence of the auto-correlation tensor.

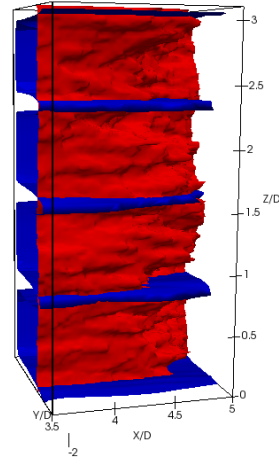
The regions of the flow affected by the auto-correlation term can be characterised by plotting the contours of SGS dissipation density $((\nabla \mathbf{u})\mathbf{a}(\nabla \mathbf{u})^T)$ of the MULU averaged in time and along the spanwise direction. These have been compared with the dissipation densities for the Smag and DSmag models $((\nabla \mathbf{u})\nu_t(\nabla \mathbf{u})^T)$ (see figure 17). A ‘reference’ dissipation density has been obtained by filtering the DNS dissipation density at the cLES resolution (see figure 16e) and by plotting the difference. The StSp model density matches best with the DNS compared with all other models - a larger spatial extent and a better magnitude match for the dissipation density is observed. The high dissipation density observed just behind the recirculation zone is captured only by the StSp model while all Smag models under-predict the density in this region. The longer recirculation zone for Smag model can be observed in the density contours. A few important questions need to be addressed here: firstly, the Smag model is known to be over-dissipative, however, in the density contours, a lower magnitude is observed for this model. This is a case of cause and effect where the over-dissipative nature of the Smag model smooths the velocity field thus reducing the velocity gradients which inversely affects the value of the dissipation density. Secondly, in the statistical comparison only a marginal difference is observed especially between the DSmag and StSp models while in the dissipation density contours we observe considerable difference. This is because the statistical profiles are a result of contributions from the resolved scales and the sub-grid scales. The dissipation density contours of figure 17 represent only a contribution of the sub-grid scales, i.e. the scales of turbulence characterised by the model. Thus, larger differences are observed in this case due to focus on the scales of model activity. Finally, we observe in figure 9 that within the vortex bubbles behind the cylinder the MULU perform better than the Smag or DSmag models. For the StSp model, this improvement is associated with the higher magnitude seen within this region in the SGS dissipation density.



(a) Full scale view of the isocontour superposition with the outlined zoom area



(b) Zoomed in view of the DNS mean vorticity isocontour



(c) Zoomed in view of the isocontour superposition

Figure 15: 3D isocontour superposition of the mean curl of $\mathbf{u}^*$ (blue) for the StSp model at $\nabla \times (\mathbf{u}^*) = 0.05$ with the mean vorticity (yellow) for DNS at $\Omega = 3$

For the StSm model, no such direct relation can be made with the SGS dissipation density. However, when we look at the resolved scale dissipation $((\nabla \mathbf{u})\nu(\nabla \mathbf{u})^T)$ for the models (see figure 16), a higher density is observed in the vortex bubbles for this model. For the classical models high dissipation is observed mainly in the shear layers. As the kinematic viscosity (ν) for all models is the same, the density maps are indicative of the smoothness of the velocity gradient. For the classical models, we see a highly smoothed field

while for the MULU, we see higher density in the wake. This difference could induce the isocontour mismatch seen in figure 9. These results are consistent with the findings of (19), who applied the MULU in the context of reduced order model and observed that MULU plays a significant role in the very near wake where important physical mechanisms take place.

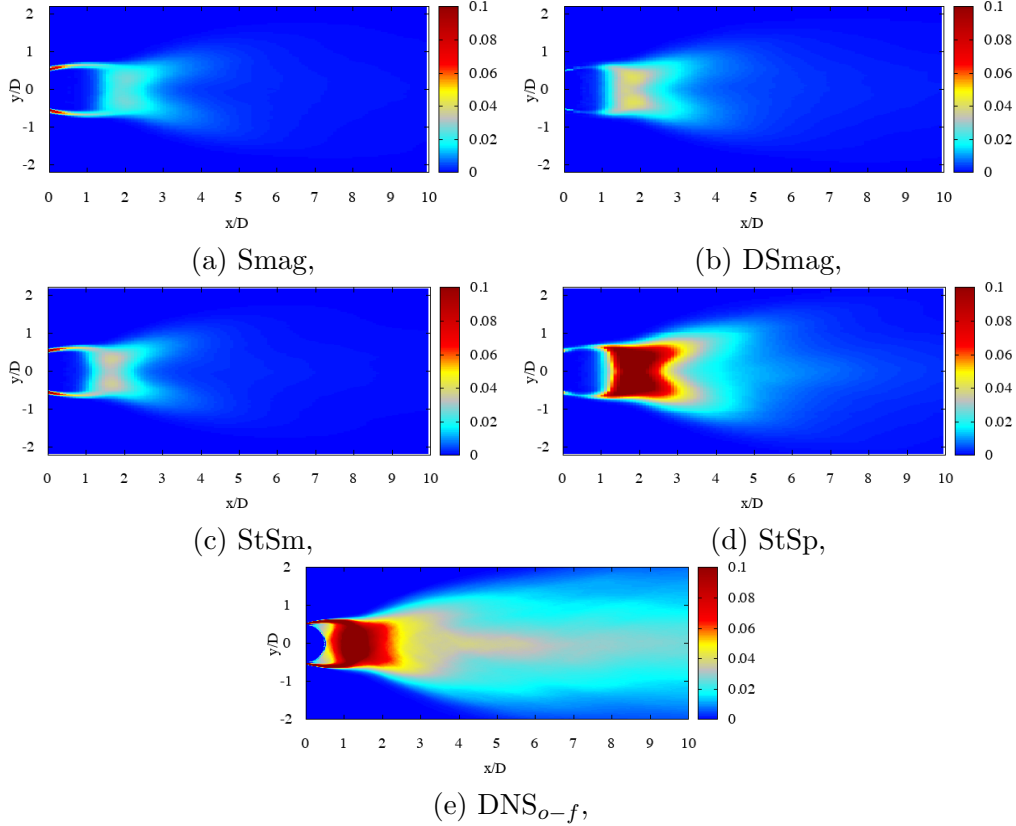


Figure 16: Sub-grid scale dissipation density in the wake of the cylinder. o stands for the original DNS dissipation and f stands for filtered (to cLES resolution) DNS dissipation.

For the MULU, the SGS contributions can be split into velocity bias ($\mathbf{u} \nabla^T (-\frac{1}{2} \nabla \cdot \mathbf{a})$) and dissipation ($\frac{1}{2} \sum_{ij} \partial_{x_i} (a_{ij} \partial_{x_j} \mathbf{u})$). Figure 18 shows the contribution of the two via 3D isocontours (dissipation in yellow and velocity bias in red). The contribution of velocity bias is limited for the StSm model as expected while in the StSp model it plays a larger role. Velocity bias in StSp model is dominant in the near wake of the flow especially in and around the recirculation zone. It is important to outline that this is the

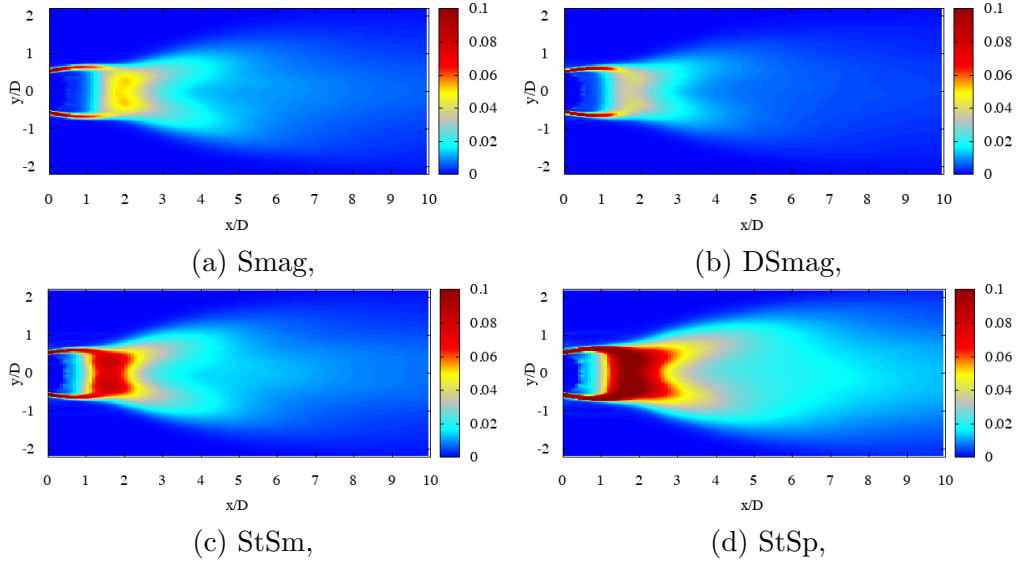


Figure 17: Resolved scale dissipation density in the wake of the cylinder for each model.

region where a statistical mismatch is observed for Smag statistics which are possibly corrected by the velocity bias in the StSp model.

Under the context of cLES, the statistical accuracy and stability of the MULU have been established with this study to be comparable, if not better, than the dynamic Smagorinsky model. The aim of performing cLES is to reduce the computational cost of simulations and hence opening newer avenues of research. This necessitates a study of the computational requirement for each model (see table 4). All simulations were performed with the same hardware and the simulation time, presented in table 4, is the time per iteration for each model averaged over two simulations of 50 time-steps each. W/o model LES or under resolved DNS is the fastest as there is no extra model calculations involved. For the cLES, MULU are computationally more expensive than classic Smagorinsky model as they involve calculation of both SGS dissipation and the velocity bias terms - this leads to a marginally increased cost for the StSm model compared to the Smag model. The StSp model is roughly twice (~ 1.98) as expensive as Smag models due to statistical variance calculations for $\mathbf{a}$ performed over a local neighbourhood. However, the slight increase in computational cost is off-set by the statistical improvements of the StSp model over the Smag model. The cost of performing the StSp model is 37% the cost of the DSmag model. This highlights

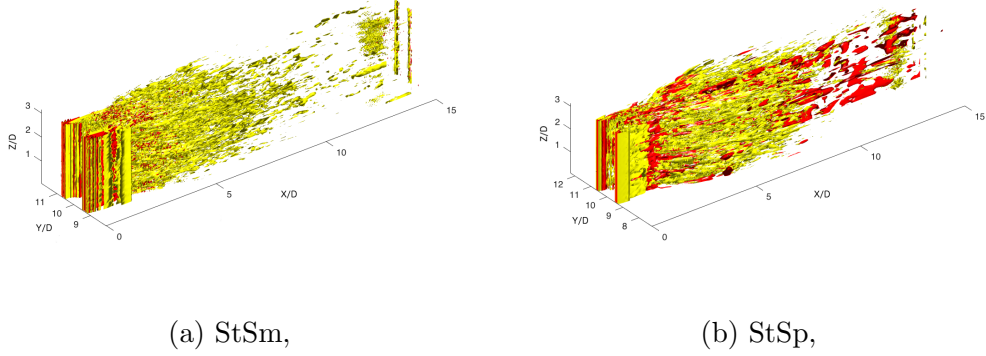


Figure 18: 3d SGS contribution iso-surface along the primary flow direction (x) with dissipation iso-surface in yellow (at 0.002) and velocity bias in red (at 0.001)

Table 4: Computational cost [s] per iteration for each model.

Model	w/o model	Smag	DSmag	StSm	StSp
Computational cost [s/iteration]	1.34	2.877	15.15	3.654	5.711

the most important feature of the StSp model - the model captures accurately the statistics at only 0.37 the cost of performing the DSmag model. However, it is important to note that while the computational cost for the Smagorinsky models stay fixed despite changes in model parameters, the cost for the StSp model strictly depends on the size of the local neighbourhood used. A smaller neighbourhood reduces the simulation cost but could lead to loss of accuracy and vice versa for a larger neighbourhood. The definition of an optimal local neighbourhood is one avenue of future research that could be promising. StSm model, which also provides comparable improvement on the classic Smag model, can be performed at 24% the cost of the DSmag model. Thus, the MULU provide a low cost (2/3rd reduction) alternative to the dynamic Smagorinsky model while improving the level of statistical accuracy.

5. Conclusion

In this study, cylinder wake flow in the transitional regime was simulated in a coarse mesh construct using the formulation under location uncertainty. The simulations were performed on a mesh 54 times coarser than the DNS study. The simulation resolution is of comparable size with PIV resolution - this presents a useful tool for performing DA where in disparity between the two resolutions can lead to difficulties.

This study focused on the MULU whose formulation introduces a velocity bias term in addition to the SGS dissipation term. These models were compared with the classic and dynamic Smagorinsky model. The MULU were shown to perform well with a coarsened mesh - the statistical accuracy of the spatial variance based model was, in general, better than the other compared models. The spatial variance based model and DSmag model both captured accurately the volatile recirculation length. The 2D streamwise velocity isocontours of the MULU matched better with the DNS reference than the Smagorinsky models. Additionally, the physical characterisation of the MULU showed that the velocity bias improved the statistics - considerably in the case of the StSp model. The analogy of the velocity bias with velocity was explored further through divergence and curl functions. The spanwise periodicity observed at low Re in literature was observed at this higher Re with the StSp model through the curl of $\mathbf{u}^*$ (analogous with vorticity) and through mean vorticity albeit noisily. The SGS contribution was compared with Smagorinsky models and the split of the velocity bias and dissipation was also delineated through isocontours.

The authors show that the performance of the MULU under a coarse mesh construct could provide the necessary computational cost reduction needed for performing LES of higher Re flows. The higher cost of the StSp model compared with Smagorinsky, is compensated by the improvement in accuracy obtained at coarse resolution. In addition, the StSp model performs marginally better than the currently established DSmag model at just 37% the cost of the DSmag model. This cost reduction could pave the way for different avenue of research such as sensitivity analyses, high Reynolds number flows, etc. Of particular interest is the possible expansion of Data Assimilation studies from the currently existing 2D assimilations (14) or low Re 3D assimilations (58) to a more informative 3D assimilation at realistic Re making use of advanced experimental techniques such as tomo-PIV. Also and more importantly, the simplistic definition of the MULU facilitate an

easy variational assimilation procedure.

6. Acknowledgements

The authors would like to thank EPSRC for the computational time made available on the UK supercomputing facility ARCHER via the UK Turbulence Consortium (EP/L000261/1).

7. Bibliography

- [1] A. A. Townsend, Momentum and Energy Diffusion in the Turbulent Wake of a Cylinder, Proc. Math. Phys. Eng. Sci. 197 (1048) (1949) 124–140. doi:10.1098/rspa.1949.0054.
URL <http://rspa.royalsocietypublishing.org/cgi/doi/10.1098/rspa.1949.0054>
- [2] A. A. Townsend, The fully developed wake of a circular cylinder, Aust. J. Chem. 2 (4) (1949) 451–468.
- [3] S. Singh, S. Mittal, Energy spectra of flow past a circular cylinder, Int. J. Comput. Fluid Dyn. 18 (8) (2004) 671–679.
URL <http://www.tandfonline.com/doi/abs/10.1080/10618560410001730278>
- [4] A. Kravchenko, P. Moin, Numerical studies of flow over a circular cylinder at $Re_D=3900$, Phys. Fluids 12 (2) (2000) 403. doi:10.1063/1.870318.
URL <http://scitation.aip.org/content/aip/journal/pof2/12/2/10.1063/1.870318>
- [5] S. Dong, G. Karniadakis, DNS of flow past a stationary and oscillating cylinder at, J. Fluids Struct. 20 (4) (2005) 519–531. doi:10.1016/j.jfluidstructs.2005.02.004.
URL <http://linkinghub.elsevier.com/retrieve/pii/S0889974605000381>
- [6] X. Ma, G. Karamanos, G. Karniadakis, Dynamics and low-dimensionality of a turbulent near wake, J. Fluid Mech. 410 (2000) 29–65.
URL http://journals.cambridge.org/abstract_S0022112099007934

- [7] P. Parnaudeau, J. Carlier, D. Heitz, E. Lamballais, Experimental and numerical studies of the flow over a circular cylinder at Reynolds number 3900, *Phys. Fluids* 20 (8) (2008) 085101. doi:10.1063/1.2957018. URL <http://scitation.aip.org/content/aip/journal/pof2/20/8/10.1063/1.2957018>
- [8] P. Beaudan, P. Moin, Numerical experiments on the flow past a circular cylinder at sub-critical Reynolds number, Stanford Univ. Ca Thermosciences Div.
- [9] L. M. Lourenco, C. Shih, Characteristics of the plane turbulent near wake of a circular cylinder, A particle image velocimetry study.
- [10] R. Mittal, P. Moin, Suitability of Upwind-Biased Finite Difference Schemes for Large-Eddy Simulation of Turbulent Flows, *AIAA Journal* 35 (8) (1997) 1415–1417. doi:10.2514/2.253. URL <http://arc.aiaa.org/doi/abs/10.2514/2.253>
- [11] J. Smagorinsky, General circulation experiments with the primitive equations, *Mon. Weather Rev.* 91 (3) (1963) 99–164. URL <http://www.ewp.rpi.edu/hartford/~ferraj7/ET/Other/References/Gawronski/LES/Smagorinsky1963.pdf>
- [12] J. Meyers, P. Sagaut, On the model coefficients for the standard and the variational multi-scale Smagorinsky model, *J. Fluid Mech.* 569 (2006) 287. doi:10.1017/S0022112006002850. URL http://www.journals.cambridge.org/abstract_S0022112006002850
- [13] B. Combès, D. Heitz, A. Guibert, E. Mémin, A particle filter to reconstruct a free-surface flow from a depth camera, *Fluid Dyn. Res.* 47 (5) (2015) 051404. URL <http://iopscience.iop.org/article/10.1088/0169-5983/47/5/051404/meta>
- [14] A. Gronskis, D. Heitz, E. Mémin, Inflow and initial conditions for direct numerical simulation based on adjoint data assimilation, *J. Comput. Phys.* 242 (2013) 480–497. doi:10.1016/j.jcp.2013.01.051. URL <http://linkinghub.elsevier.com/retrieve/pii/S0021999113001290>

- [15] Y. Yang, C. Robinson, D. Heitz, E. Mémin, Enhanced ensemble-based 4dvar scheme for data assimilation, *Comput. Fluids* 115 (2015) 201–210.
doi:10.1016/j.compfluid.2015.03.025.
URL <http://linkinghub.elsevier.com/retrieve/pii/S0045793015000985>
- [16] E. Mémin, Fluid flow dynamics under location uncertainty, *Geophys. Astrophys. Fluid Dynam.* 108 (2) (2014) 119–146.
doi:10.1080/03091929.2013.836190.
URL <http://www.tandfonline.com/doi/abs/10.1080/03091929.2013.836190>
- [17] V. Resseguier, E. Mémin, B. Chapron, Geophysical flows under location uncertainty, part I: Random transport and general models, *Geophys. Astrophys. Fluid Dynam.* 11 (3) (2017) 149–176.
- [18] S. K. Harouna, E. Mémin, Stochastic representation of the Reynolds transport theorem: revisiting large-scale modeling, *Comput. Fluids* doi:10.1016/j.compfluid.2017.08.017.
URL <http://linkinghub.elsevier.com/retrieve/pii/S0045793017302797>
- [19] V. Resseguier, E. Mémin, D. Heitz, B. Chapron, Stochastic modelling and diffusion modes for proper orthogonal decomposition models and small-scale flow analysis, *J. Fluid Mech.* 826 (2017) 888–917.
doi:10.1017/jfm.2017.467.
URL https://www.cambridge.org/core/product/identifier/S0022112017004670/type/journal_article
- [20] A. Bensoussan, R. Temam, Equations stochastiques du type Navier-Stokes, *J. Funct. Anal.* 13 (2) (1973) 195–222.
- [21] P. J. Mason, D. J. Thomson, Stochastic backscatter in large-eddy simulations of boundary layers, *J. Fluid Mech.* 242 (1992) 51–78.
- [22] U. Schumann, Stochastic backscatter of turbulence energy and scalar variance by random subgrid-scale fluxes, *Proc. R. Soc. A* 451 (1995) 293–318.
- [23] R. H. Kraichnan, The structure of isotropic turbulence at very high Reynolds numbers, *J. Fluid Mech.* 5 (04) (1959) 497–543.

URL http://journals.cambridge.org/abstract_S0022112059000362

- [24] C. Leith, Atmospheric predictability and two-dimensional turbulence, *J. Atmos. Sci.* 28 (2) (1971) 145–161.
- [25] J. P. Laval, B. Dubrulle, J. C. McWilliams, Langevin models of turbulence: Renormalization group, distant interaction algorithms or rapid distortion theory?, *Phys. Fluids* 15 (5) (2006) 1327–1339.
- [26] R. Kraichnan, Convergents to turbulence functions, *J. Fluid Mech.* 41 (1970) 189–217.
- [27] B. Sawford, Generalized random forcing in random-walk models of turbu-, *Phys. Fluids* 29 (1986) 3582–3585.
- [28] D. Haworth, S. Pope, A generalized langevin model for turbulent flows., *Phys. Fluids* 29 (1986) 387–405.
- [29] S. Pope, Lagrangian pdf methods for turbulent flows, *Annu. Rev. Fluid Mech* 26 (1994) 23–63.
- [30] S. Pope, *Turbulent flows*, Cambridge University Press, 2000.
- [31] J. M. MacInnes, F. V. Bracco, Stochastic particle dispersion modeling and the tracer-particle limit, *Phys. Fluids A: Fluid Dyn.* 4 (12) (1992) 2809. doi:10.1063/1.858337.
URL <http://scitation.aip.org/content/aip/journal/pofa/4/12/10.1063/1.858337>
- [32] H. Kunita, *Stochastic Flows and Stochastic Differential Equations*, Cambridge University Press, 1990.
- [33] V. Resseguier, E. Mémin, B. Chapron, Geophysical flows under location uncertainty, Part II: Quasigeostrophic models and efficient ensemble spreading, *Geophys. Astrophys. Fluid Dynam.* 11 (3) (2017) 177–208.
- [34] V. Resseguier, E. Mémin, B. Chapron, Geophysical flows under location uncertainty, Part III: SQG and frontal dynamics under strong turbulence, *Geophys. Astrophys. Fluid Dynam.* 11 (3) (2017) 209–227.

- [35] D. D. Holm, Variational principles for stochastic fluid dynamics, *Proc. Math. Phys. Eng. Sci.* 471 (2176) (2015) 20140963–20140963. doi:10.1098/rspa.2014.0963.
URL <http://rspa.royalsocietypublishing.org/cgi/doi/10.1098/rspa.2014.0963>
- [36] D. Crisan, F. Flandoli, D. D. Holm, Solution properties of a 3d stochastic Euler fluid equation, *arXiv preprint arXiv:1704.06989*.
URL <https://arxiv.org/abs/1704.06989>
- [37] C. J. Cotter, G. A. Gottwald, D. D. Holm, Stochastic partial differential fluid equations as a diffusive limit of deterministic Lagrangian multi-time dynamics, *arXiv preprint arXiv:1706.00287*.
URL <https://arxiv.org/abs/1706.00287>
- [38] D. K. Lilly, The representation of small scale turbulence in numerical simulation experiments, *Proc. IBM Sci. Compt. Symp. Environ. Sci.* (1967) 195–210.
- [39] M. Germano, U. Piomelli, P. Moin, W. Cabot, A dynamic subgrid-scale eddy viscosity model, *Phys. Fluids A: Fluid Dyn.* 3 (7) (1991) 1760. doi:10.1063/1.857955.
URL <http://scitation.aip.org/content/aip/journal/pofa/3/7/10.1063/1.857955>
- [40] E. R. Van Driest, The problem of aerodynamic heating, *Aero. Engg. Rev.* 15 (10) (1956) 26–41.
- [41] V. Yakhot, S. A. Orszag, Renormalization group analysis of turbulence. I. Basic theory, *J. Sci. Comput.* 1 (1) (1986) 3–51.
URL <http://link.springer.com/article/10.1007/BF01061452>
- [42] M. Breuer, N. Jovičić, K. Mazaev, Comparison of DES, RANS and LES for the separated flow around a flat plate at high incidence, *Int. J. Numer. Meth. Fluids* 41 (4) (2003) 357–388.
URL <http://onlinelibrary.wiley.com/doi/10.1002/flid.445/full>
- [43] H. Ouvrard, B. Koobus, A. Dervieux, M. V. Salvetti, Classical and variational multiscale LES of the flow around a circular cylinder

- on unstructured grids, *Comput. Fluids* 39 (7) (2010) 1083–1094.
doi:10.1016/j.compfluid.2010.01.017.
URL <http://linkinghub.elsevier.com/retrieve/pii/S0045793010000319>
- [44] S. Laizet, E. Lamballais, High-order compact schemes for incompressible flows: A simple and efficient method with quasi-spectral accuracy, *J. Comput. Phys.* 228 (16) (2009) 5989–6015.
doi:10.1016/j.jcp.2009.05.010.
URL <http://linkinghub.elsevier.com/retrieve/pii/S0021999109002587>
- [45] S. Lele, Compact finite difference schemes with spectral-like resolution, *J. Comput. Phys.* 103 (1) (1992) 16–42.
- [46] R. Gautier, S. Laizet, E. Lamballais, A DNS study of jet control with microjets using an immersed boundary method, *Int. J. Comput. Fluid Dyn.* 28 (6-10) (2014) 393–410. doi:10.1080/10618562.2014.950046.
URL <http://www.tandfonline.com/doi/abs/10.1080/10618562.2014.950046>
- [47] L. C. Pinto, E. B. C. Schettini, J. H. Silvestrini, Numerical analysis of the immersed boundary method applied to the flow around a forced oscillating cylinder, *J. Phys. Conf. Ser.* 296 (2011) 012011.
doi:10.1088/1742-6596/296/1/012011.
URL <http://stacks.iop.org/1742-6596/296/i=1/a=012011?key=crossref.596e99f672722522c1ffb573038e4aff>
- [48] S. Laizet, N. Li, Incompact3d: A powerful tool to tackle turbulence problems with up to $O(10^5)$ computational cores, *Int. J. Numer. Meth. Fluids* 67 (11) (2011) 1735–1757. doi:10.1002/fld.2480.
URL <http://doi.wiley.com/10.1002/fld.2480>
- [49] O. Métais, M. Lesieur, Spectral large-eddy simulation of isotropic and stably stratified turbulence, *J. Fluid Mech.* 239 (1992) 157.
doi:10.1017/S0022112092004361.
URL http://www.journals.cambridge.org/abstract_S0022112092004361

- [50] C. Norberg, LDV measurements in the near wake of a circular cylinder, in: Proceedings of the Conference on Bluff Body Wakes and Vortex-Induced Vibration, Cornell University, Ithaca, NY, 1998, Washington, D.C., 1998, pp. 1–12.
- [51] T. Corpetti, D. Heitz, G. Arroyo, E. Mmin, A. Santa-Cruz, Fluid experimental flow estimation based on an optical-flow scheme, *Experiments in Fluids* 40 (1) (2006) 80–97. doi:10.1007/s00348-005-0048-y.
URL <http://link.springer.com/10.1007/s00348-005-0048-y>
- [52] L. Ong, J. Wallace, The velocity field of the turbulent very near wake of a circular cylinder, *Exp. Fluids* 20 (6) (1996) 441–453.
URL <http://link.springer.com/article/10.1007/BF00189383>
- [53] C. H. K. Williamson, Vortex dynamics in the cylinder wake, *Annu. Rev. Fluid Mech.* 28 (1996) 477–539.
- [54] C. Chyu, D. Rockwell, Near-wake structure of an oscillating cylinder: effect of controlled shear-layer vortices, *J. Fluid Mech.* 322 (1996) 21–49.
URL http://journals.cambridge.org/abstract_S0022112096002698
- [55] M. Lesieur, P. Comte, Y. Dubief, E. Lamballais, O. Métais, S. Ossia, From two-point closures of isotropic turbulence to LES of shear flows, *Flow Turbul. Combust.* 63 (1-4) (2000) 247–267.
URL <http://link.springer.com/article/10.1023/A:1009900723783>
- [56] F. Ducros, P. Comte, M. Lesieur, Large-eddy simulation of transition to turbulence in a boundary layer developing spatially over a flat plate, *J. Fluid Mech.* 326 (1996) 1–36.
- [57] B. Bays-Muchmore, A. Ahmed, On streamwise vortices in turbulent wakes of cylinders, *Phys. Fluids A: Fluid Dyn.* 5 (2) (1993) 387–392. doi:10.1063/1.858862.
URL <http://aip.scitation.org/doi/10.1063/1.858862>
- [58] C. Robinson, Image assimilation techniques for Large Eddy Scale models : Application to 3d reconstruction, (doctoral thesis, Université de Rennes 1, Rennes (2015)).